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#### Case Study on Coupling Crack on a Centrifugal Pump

Ahmed Ashour Ismail

Sherif Mekawey

**Chris Shepherd** 

### Authors



#### Ahmed Ashour Ismail

Lead Machinery Diagnostics Engineer at BH-Bently Nevada Machinery Diagnostics Services in Middle East. 14 years condition monitoring and vibration analysis experience, working for Bently Nevada since 2014. Assigned as Site Service Agreement focal point for British Petroleum. Certified CAT-3 Vibration Analyst, CAT-1 Thermography, CMRP & CRL,.

#### Sherif Mekawey- Principal Engineer



Chris Shepherd

The Technical Leader of BH-Bently Nevada Machinery Diagnostics Services for Middle East, North Africa (MENA) Region. Working for BH-Bently Nevada for 25 years and has over 35 years experience in vibration field, including rotating equipment balancing, condition monitoring, vibration analysis, diagnostics, and root cause analysis. Published several case studies in Orbit magazines and Turbomachinery symposiums.

Senior Rotating Equipment Engineer at BP, 25 Years Condition Monitoring & Vibration Analyst & Reliability & Rotating Equipment and Site Operations and Maintenance Roles, CAT3 Certified Vibration Analyst, certified Motion Amplification Analyst.

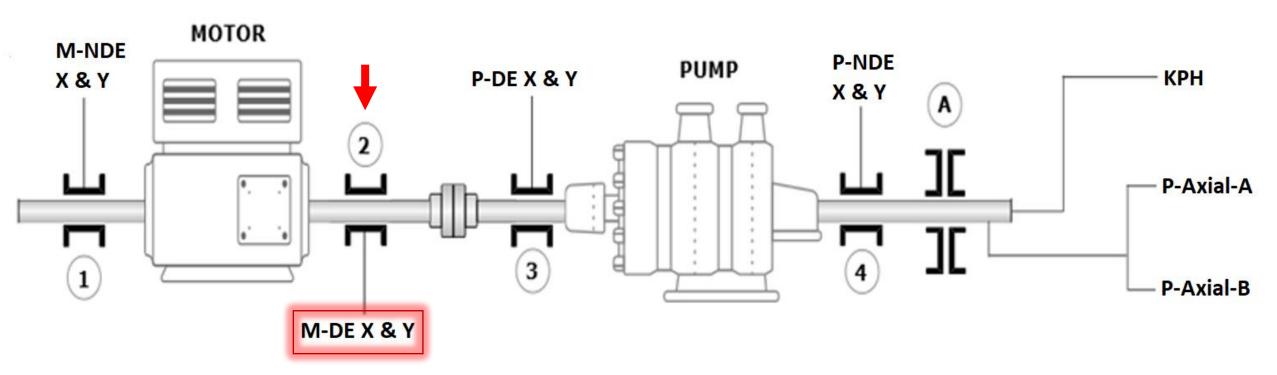


### Abstract

- This case study is focused on accurately identifying the root cause of the observed change in the rotor natural frequency by analyzing the vibration excursions on a motor driving a centrifugal pump through a flexible coupling.
- The unit tripped on high vibration at Motor DE bearing probes, This trip event had no records on the online monitoring server due to communication loss between the software and the Pump Monitoring Software.
- However, the machinery protection system was able to trip the unit successfully. The communication loss issue was resolved, and the unit started back up and kept running for approximately 20 minutes until it tripped again on the same Motor DE bearing vibration probes.



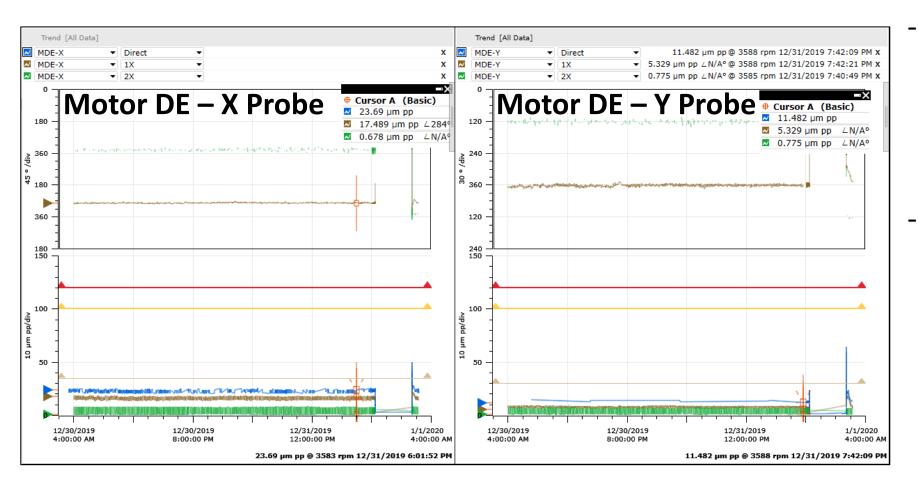
#### Machine Layout



Train consists of 5000KW/60HZ (3588 RPM) induction electric motor driving a centrifugal pump through a flexible shim-pack coupling.



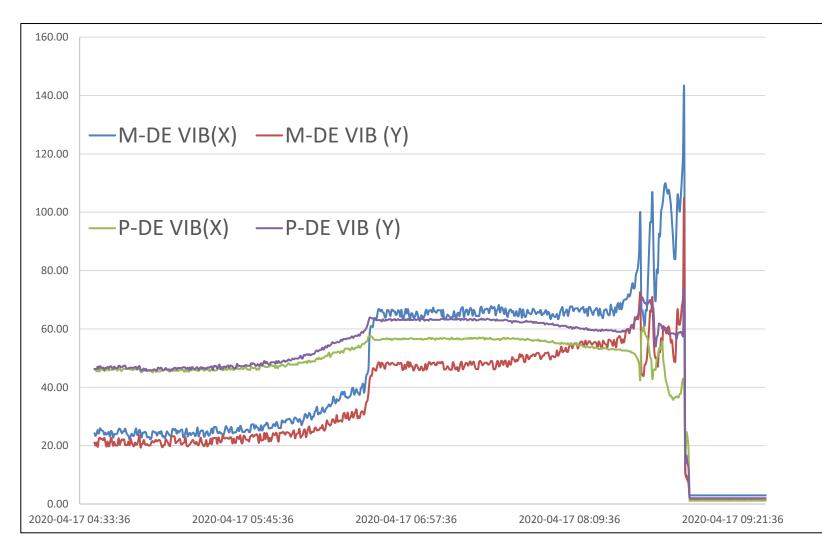
# Previous Data Analysis prior to communication loss



Motor DE vibration probes shows acceptable vibration levels before the high vibration event. Unfortunately, no vibration data were available for approximately 4 months before the high vibration event.



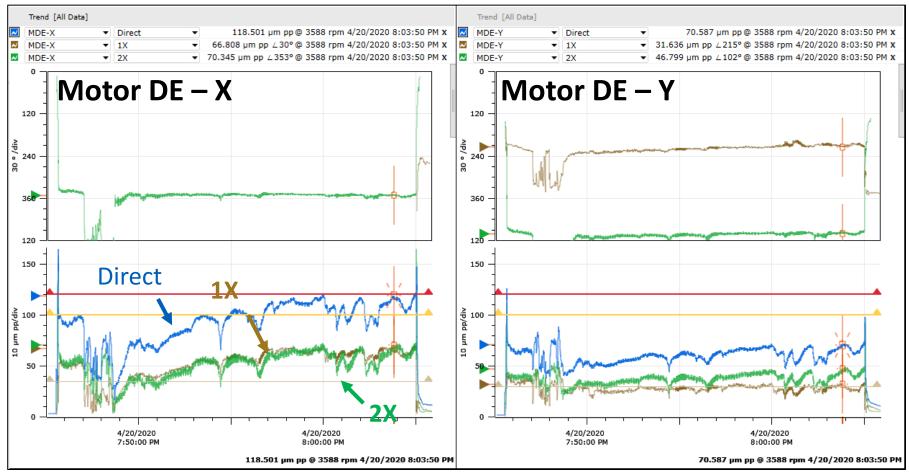
### DCS vibration trend during steady State at the vibration trip event



- DCS vibration trend show
   the gradual increase in
   vibration levels until tripping
   the unit on motor DE.
- This trip event had no records on the online monitoring server due to communication loss between the software and the pump monitoring system.

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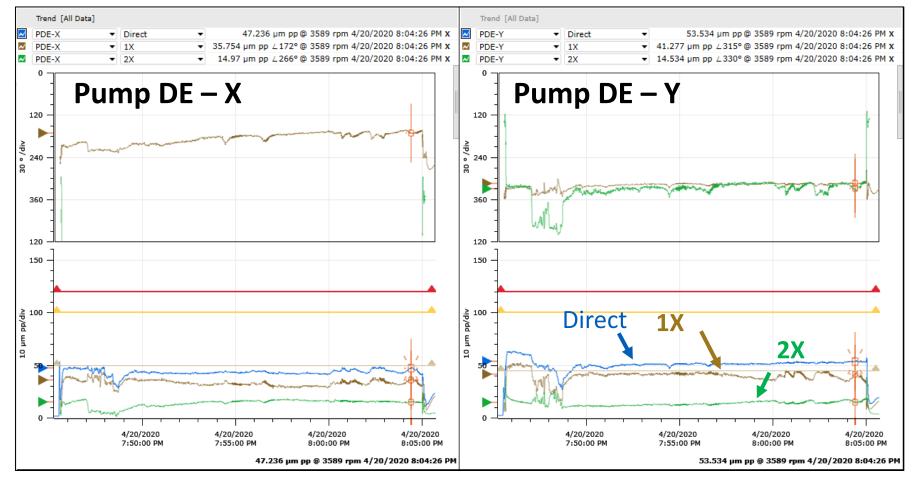
## Motor Drive-End Trends after resolving the communication issue



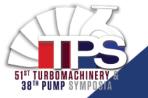
- Motor DE bearing X & Y probes vibration trends recorded on the following startup shows the dominant 1X & 2X filtered peaks raising the overall value. Also, it reveals a rapid fluctuation behavior until tripping the unit again.



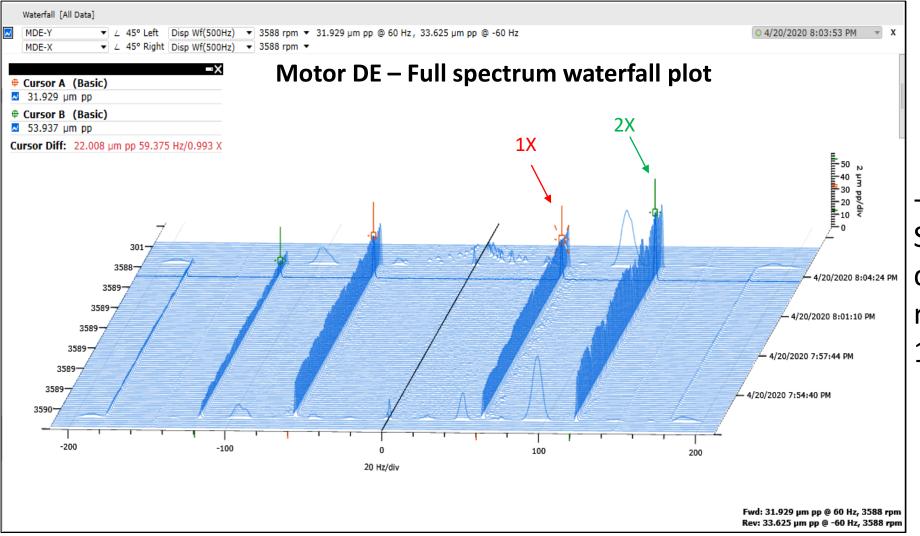
## Pump Drive-End Trends after resolving the communication issue



Pump DE bearing side
reveals the dominated 2X
peak and the fluctuation
behavior, however, the
overall values were low.



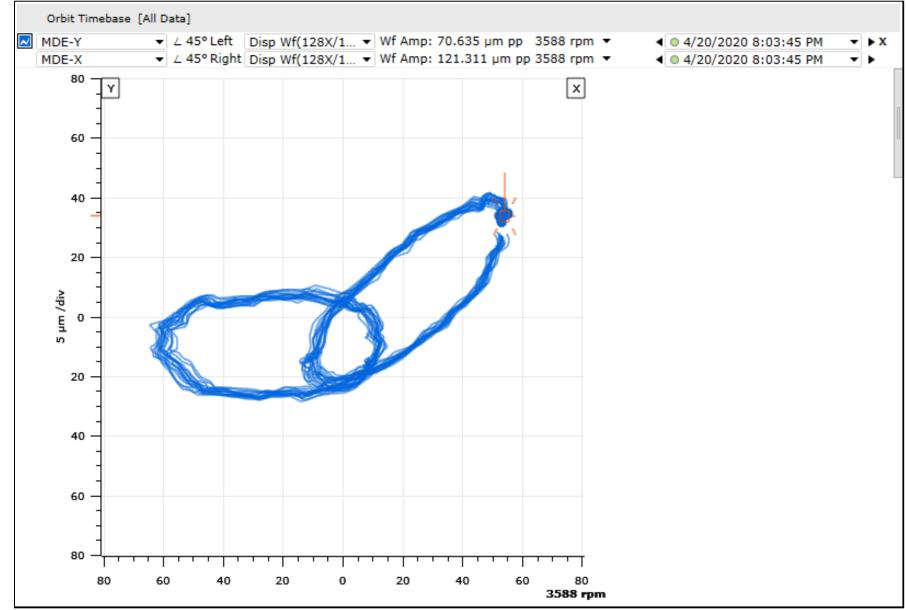
### Motor Drive-End Spectrum Analysis



Motor DE bearing FullSpectrum waterfall plot
during steady-state
reveals the dominated
1X & 2X peaks.



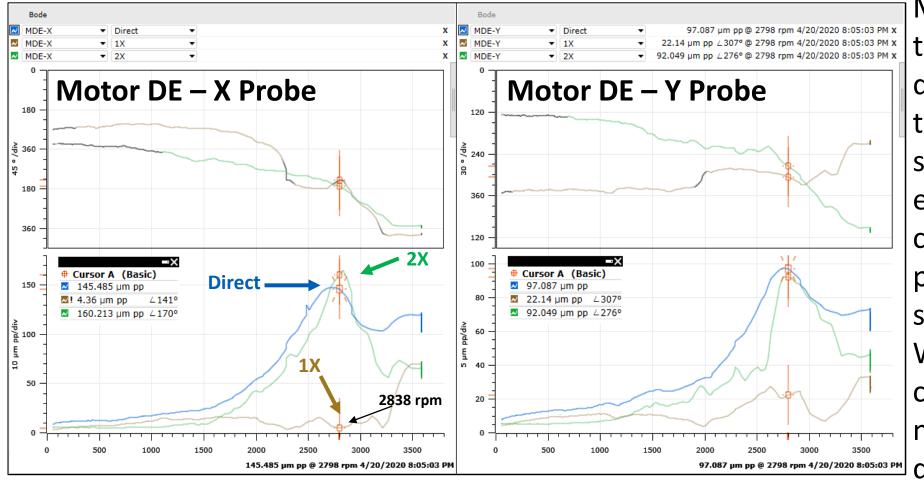
#### Motor Drive-End Orbit Analysis



Motor DE bearing orbit plot during steady-state reveals the dominated 1X & 2X peaks.

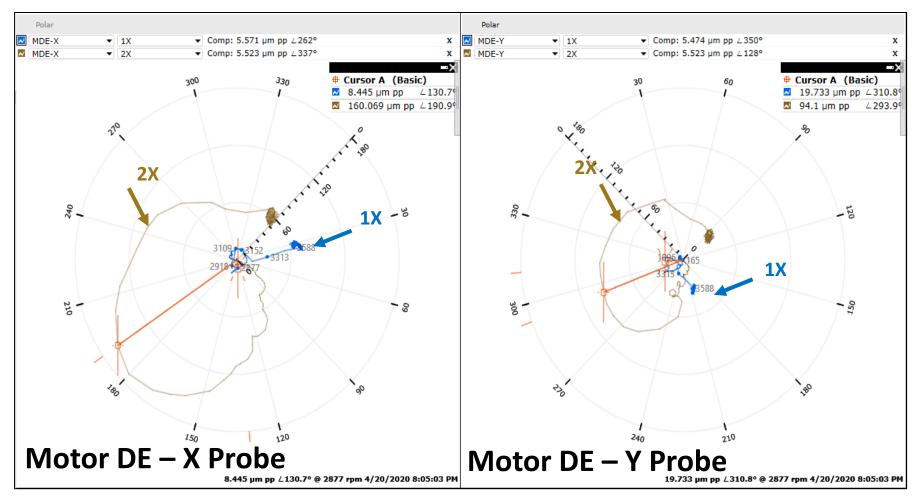


#### Transient Data Analysis – Motor DE Bode Plot



Motor DE bearing transient data analysis during shutdown throughout Bode plot show the 2X peak excitation of second critical frequency while passing half of 2nd critical speed at 2800 RPM. While 1st critical speed continues to perform normally during coasting down.

#### Transient Data Analysis – Motor DE Polar Plot

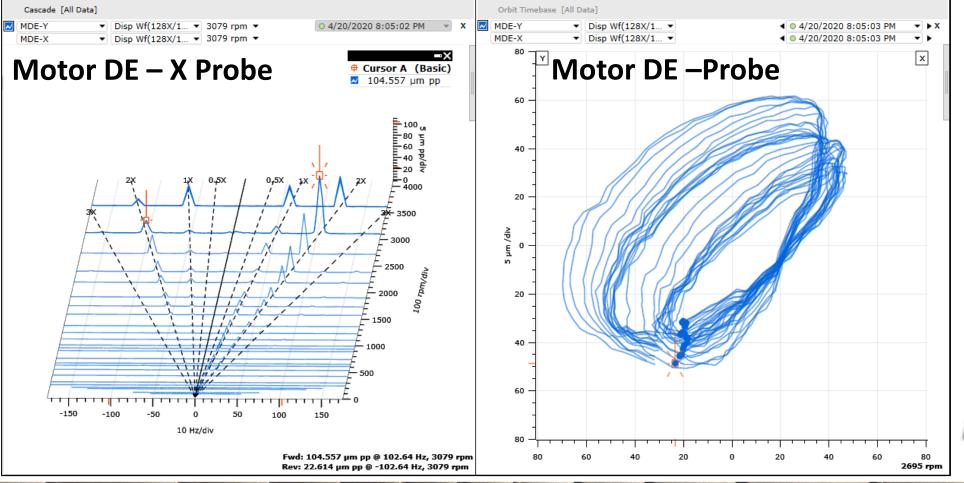


- Motor DE bearing Polar plot shows the 2X peak excitation of second critical frequency while passing half of 2nd critical speed at 2800 RPM. While the 1st critical speed continues to perform normally during coasting down.



# Transient Data Analysis – Motor DE Cascade and Orbit Plot

-Motor DE bearing synchronous Direct Orbit & Full Cascade plot while coasting down at 3231 RPM

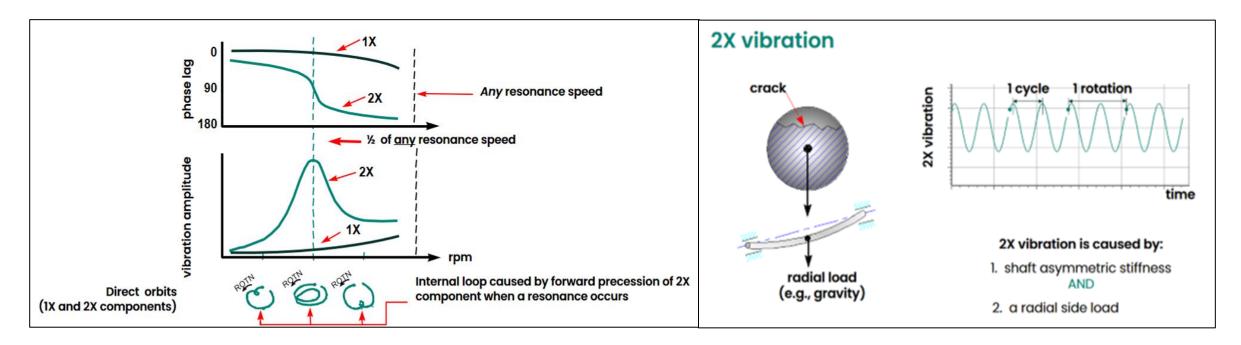


### Conclusion

- Transient data analysis throughout Bode, Polar & Full spectrum Cascade plots revealed that the 2X component exhibited abnormal behavior. While the rotor was coasting down and before passing the 1st critical speed (which is ~2300 RPM as per Motor datasheet and previously recorded data), the 2X component was excited, reaching 160  $\mu$ m PP at 2800 RPM. It appears that this 2X vibration at 2800 rpm is half of the second natural frequency of 5600 CPM as evidenced from the shutdown 2X polar plot. Thus, it is believed that the observed 2X peak belongs to excitation of the 2nd critical speed. Note that this unusual behavior was not found on previous runs.
- This phenomena represents a rotor running at a speed close to half of resonant frequency with presence of asymmetric stiffness with a radial force.



### Back to Basics: Shaft Crack Detection Rule

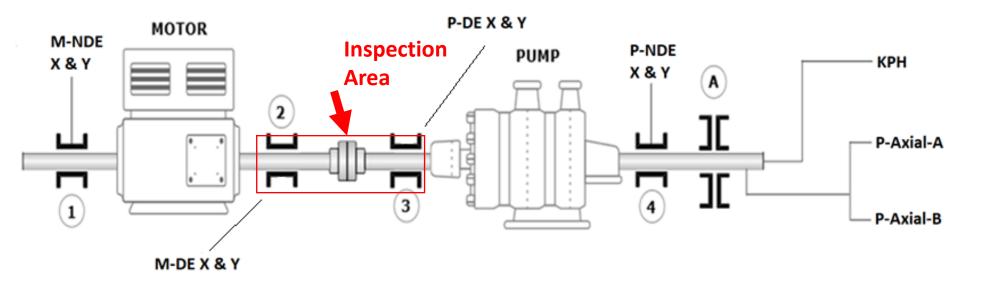


- A rotor system with an asymmetric shaft and a radial side load force, rotating at a speed near half "½" of any resonant frequency, may experience high 2X vibration amplitude and 2X phase shift.
- Other 2X vibration effects on a cracked shaft will result in a more rapid growth of the crack. This Includes increased reversal stresses & low or high cycle fatigue.



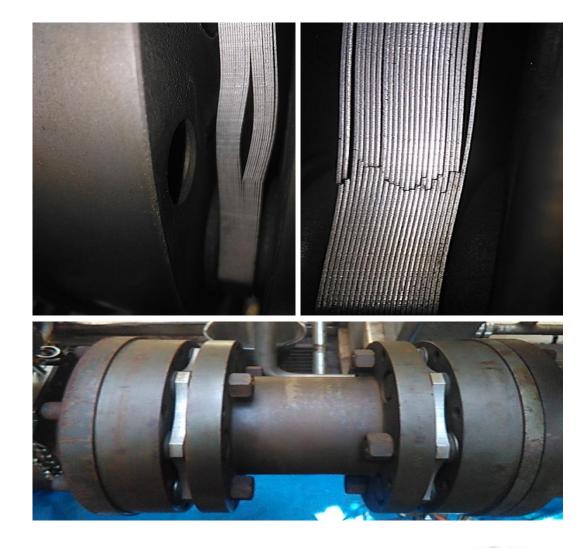
#### Recommendation

- It was concluded that the rotor is experiencing crack symptoms; hence it was recommended to:
  - Inspect the coupling between motor and pump for any sign of coupling shims damage or shaft crack.



### Findings

Upon our recommendation, the site team
 inspected the coupling and found the shims
 (membrane) are bent and broken, as shown in
 the photos and inline with what was
 recommended.





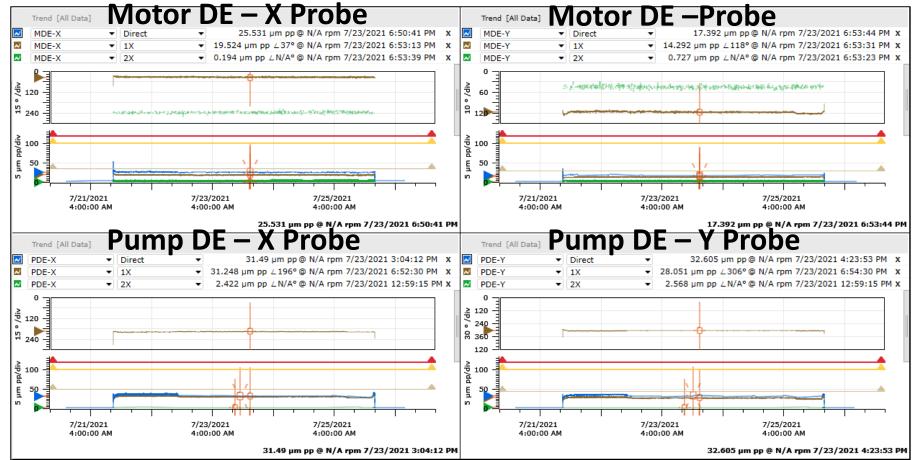
#### Lesson Learned

- Maintaining healthy communication status between the vibration monitoring software and the protection system is essential to early detect any rotor behavior deterioration.
- Software alarms were already implemented on the vibration monitoring software; however, due to the communication loss across the monitoring system and the remote monitoring team, this failure could not be detected earlier.

### Possible causes

- Historically these units are found in a highly corrosive environment in addition to corrosive foundations (another case from other sister machine) which could lead to this damage.
- The high dynamic forces coming from the motor each startup and frequently due to multiple process upsets could also contribute to experiencing this type of failure. This has been observed historically which also led to multiple pump trip events during startup.
- Minimum flow setpoint for one or two pump operation was found set incorrectly and rectified accordingly.
- Based of the findings above, asset team is working through the list of causes and eliminating them.

# Motor and Pump DE bearings trends after coupling replacement



- Coupling was replaced and the unit started again with acceptable and normal vibration levels.



# END

#### Questions?

