







## Presenter/author bios

**Zixiang Chen, Wood Group** (formerly **BETA Machinery Analysis**), PhD, EIT, Engineer – Zixiang has recently completed his doctoral work in the field of fluid dynamics. His academic research interest is on the sensor-based estimation of large-scale coherent structures in turbulent flows using optical flow diagnostics and numerical simulations. Prior to joining Wood Group in 2014, he worked on combustion modelling for the GT24/26 and compressor aerodynamics for the PT6.

Cajetan Ijeomah, Wood Group (formerly BETA Machinery Analysis), MSc, PEng, Senior Engineer

Cajetan is a professional engineer with over 14 years of industry experience, the majority of which is in dynamic engineering and vibration design. His responsibilities include acoustical (pulsation), mechanical and thermal analysis of reciprocating compressor and pump systems, evaluation of piping systems and related dynamic studies. Cajetan manages complex design projects involving multiple vibration disciplines and is proficient in the use of various engineering analysis software tools.

Jordan Grose, Wood Group (formerly BETA Machinery Analysis), PEng, MBA, Service Leader – Americas Jordan is a mechanical engineer with extensive domestic and international design, field, and monitoring experience with compressors, pumps, pipelines, and other production machinery. He has specialized skills in vibration, liquid transient analysis, performance, and troubleshooting in onshore and offshore production facilities.

Jordan currently leads Wood Group's vibration integrity group in addressing plant-wide vibration risks in piping and machinery systems. He has been with Wood Group (formerly BETA Machinery Analysis) for over 14 years, during which time he has authored and co-authored several papers. Jordan was formerly responsible for BETA's Malaysia office in Kuala Lumpur.

Kelly Eberle, Wood Group (formerly BETA Machinery Analysis), BSc, PEng, Principal Consultant

Kelly is a principal consultant and has been with Wood Group (formerly BETA Machinery Analysis) since 1988. He has accumulated a wide range of design and field experience, particularly in the area of pressure pulsation analysis and mechanical analysis of reciprocating compressor and pump installations. The scope of his experience includes pulsation simulations, thermal flexibility studies, dynamic finite element analysis, structural analysis and foundation analysis.

#### **Abstract**

It is well understood that static pressure at the inlet of reciprocating pumps, quantified typically by Net Positive Suction Head Available (NPSHA), must be sufficient to avoid cavitation in the pump suction manifold and chamber. In an effort to conserve NPSHA, pump designers generally rely on rules of thumb that resist the addition of pressure drop elements such as restrictive orifice plates, choke tubes and line-size reductions to the inlet piping of all pumps, including reciprocating pumps.

Another design consideration of reciprocating pumps is the generation of pressure pulsations due to pump piston and valve motion. Uncontrolled pulsations can result in cavitation and vibration-related fatigue failures. In many cases, pressure drop elements are required to control pressure pulsations. Can there be a balance between the pulsation control benefits of pressure drop elements and the need to meet NPSHA?

This paper is of interest to designers and engineers working with reciprocating pump installations. It aims at challenging industry resistance to using pressure drop elements in the suction piping of reciprocating pumps by, first, outlining the virtues achieved in terms of pulsation and vibration control, and second, presenting results from numerical simulations (one-dimensional pulsation and detailed CFD modelling). Recent field data from a quintuplex pump installation were used to validate the 1-D pulsation model. The results show that well-designed orifice plates, and other pressure drop elements, are beneficial for reducing pulsations and cavitation risks; and can be used in the suction piping of reciprocating pumps.



# Objective

 Demonstrate that proper use of orifice plates and other pressure drop elements (choke tubes and line-size reductions) in the suction piping of reciprocating pumps are beneficial for pulsation control and mitigation of cavitation risks.

# Analysis approach

Field-measured pulsations and numerical simulations (1-D pulsation model and CFD modelling).



#### 1. Author bios

- 2. Abstract
- 3. Contents
- 4. Objective
- 5. Introduction
- 6. 1-D model
- 7. CFD model

#### **Contents**

- 8. Validation of 1-D model
- 9. 1-D model results
- 10. Definition of pressure loss terms
- 11. Average pressure loss: 1-D vs CFD
- 12. Total pressure loss: 1-D vs CFD
- 13. CFD visualization: pressure recovery
- 14. Conclusions



## Introduction

- Reciprocating pumps:
  - Widespread use in industry
  - Generate pressure pulsations
  - Uncontrolled pulsations
    - → vibrations, cavitation, failures



- Orifice plates:
  - Reduced pressure under-spikes → lower cavitation risk
  - Pressure drop → reduced NPSHA → higher cavitation risk



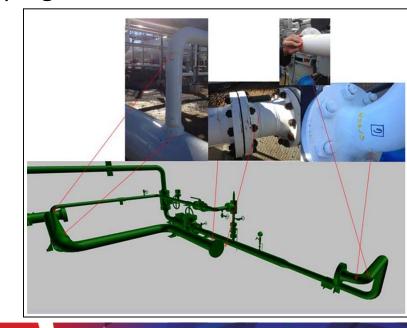


## Introduction, continued

#### Case study pump installation:

- Two 5-plunger propane pipeline pumps; 3.5" bore x 5" stroke
- -160 400 RPM, 416 GPM/pump; Ps = 98 282 psig; Pd = 1100 psig
- Elevated and inadequately supported piping
- High pulsations, vibrations and indications of cavitation
- Winter condition worse than summer due to higher liquid bulk modulus
- NDT showed several cracks in piping after six months of operation

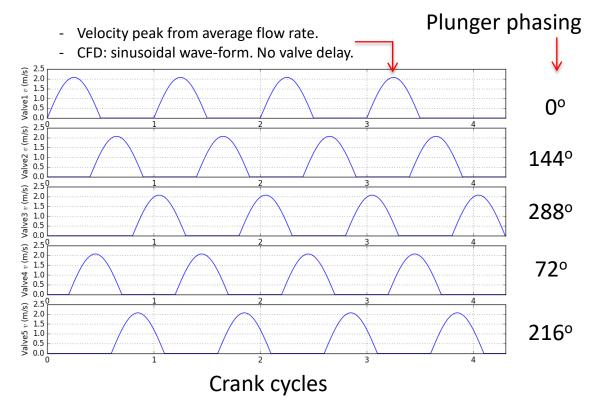




# Applied method of characteristics to solve 1-D fluid dynamics equation

- Used empirical loss factors to calculate irreversible pressure losses
- Modelled from pump valves to known boundaries in piping

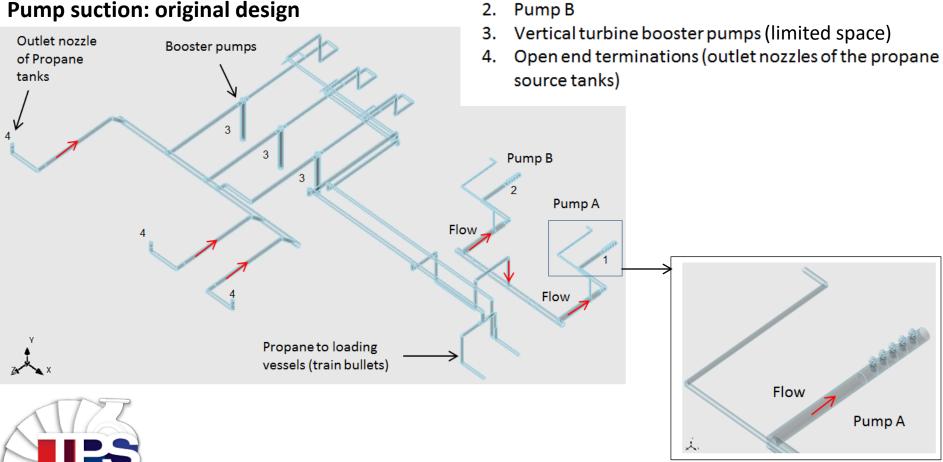
#### 1-D model



BC: Velocity profile at the pump valves

## 1-D model, continued

Pump suction: original design

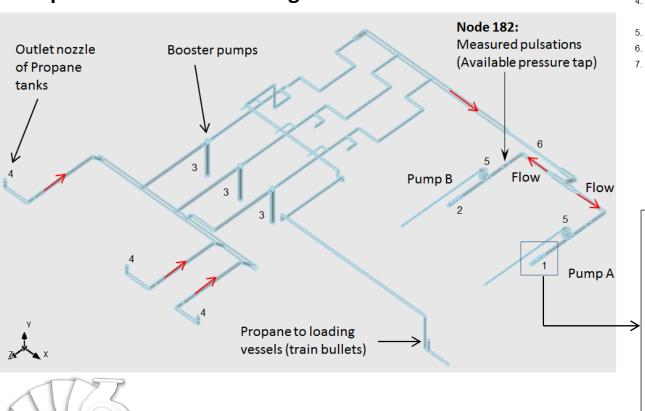


Legend:

Pump A

## 1-D model, continued

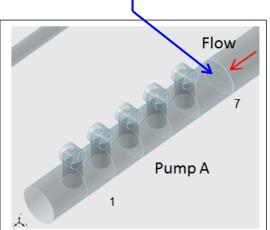
Pump suction: modified design



#### Legend:

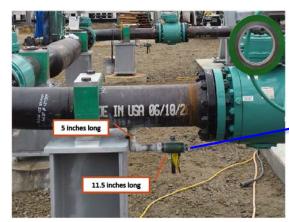
- 1. Pump A
- 2. Pump B
- 3. Vertical turbine booster pumps
- Open end terminations (outlet nozzles of the propane source tanks)
- 5. 20 gallon gas-charged dampener
- 6" Sch. 160 line reduction (external choke tube)
- 7. 3.125" ID restriction orifice plate (40% beta ratio)

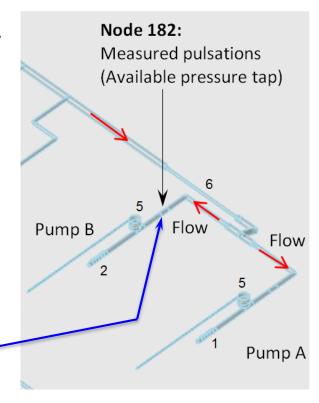




### Validation of 1-D model

- Field: pulsations measured at Node 182
- Field data vs 1-D model:
  - Field visit during summer
  - Frequency spectra at 300 RPM
  - Frequency spectra at 165 300 RPM

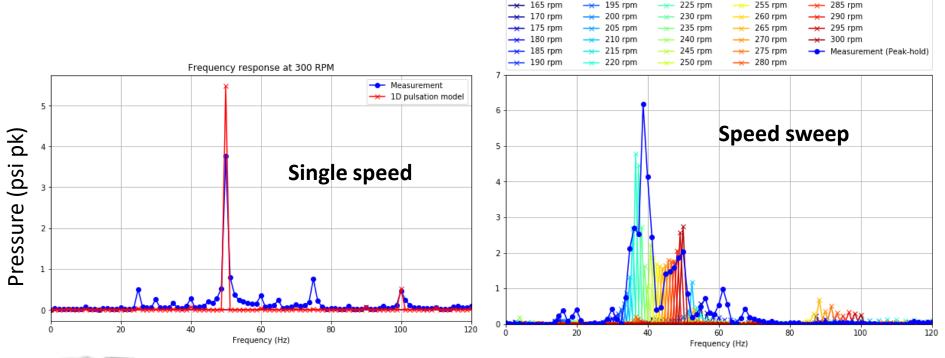






# Validation of 1-D model, continued

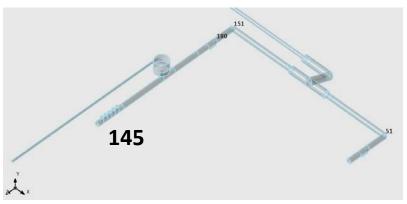
Summer condition



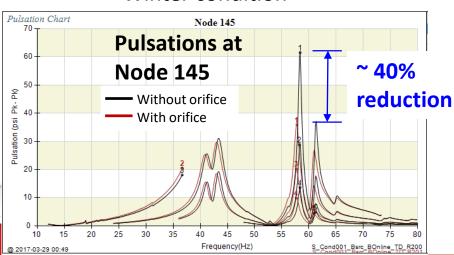


Pulsations at node 180 (without orifice plate installed)

### 1-D model results



#### Winter condition

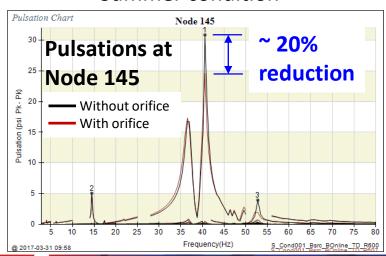


#### Key result:

Orifice plate will improve reliability.

Operator concerned with pressure drop.

#### Summer condition



#### CFD model

- Solved 3-D Navier-Stokes equation
- Modelled from pump valves to small part of suction piping
- Assumed ideal pump valve opening and closing (no valve delays)
- BC: velocity profile at valves; constant pressure at suction pipe
- Turbulence model:
  - → Large-eddy simulation (LES)
  - → Wall adapting local eddy-viscosity (WALE)
- Used reduced 1-D model for comparisons with CFD model Constant pressure (open end termination)

Sinusoidal velocity profile at pump valves





# Definition of pressure loss terms

•  $\Delta P_{avg}$ : Average pressure loss

• ΔP<sub>max</sub>: Maximum pressure loss

•  $\Delta P_{cav}$ : Pressure loss for cavitation

• P<sub>o</sub> : Static pressure at pump inlet

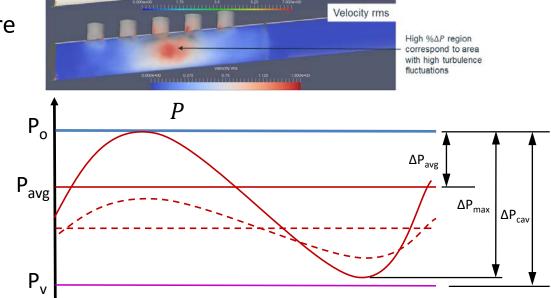
P<sub>avg</sub>: Time-average static pressure

(time: ~ 2 sec.)

P<sub>v</sub>: : Vapour pressure

Pressure without orifice plate

--- Pressure with orifice plate



Average pressure reduction

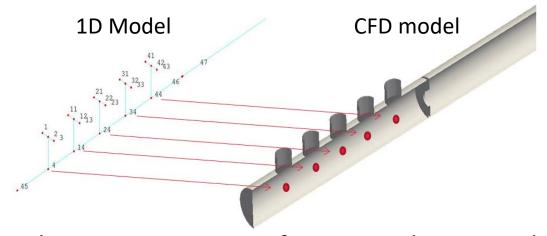
Maximum pressure reduction

far from the limit set by

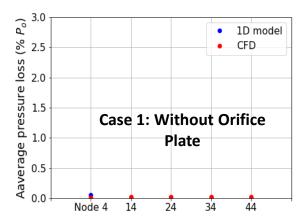
the vapour pressure

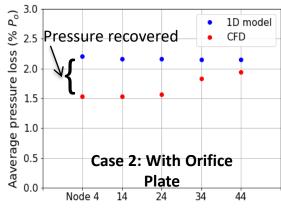


# Average pressure loss: 1-D vs CFD model



- The same geometry for 1-D and CFD models
- Average pressure loss (recoverable + irreversible)
  - → Effect of pulsations not included
  - → Higher loss with orifice plate
    - → Pressure recovery with CFD model





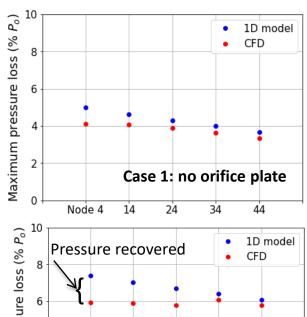
# Total pressure loss: 1-D vs CFD model

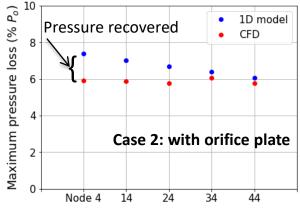
- Maximum pressure loss (pulsations + recoverable + irreversible)
  - → Mainly due to pulsations
  - → Higher with orifice plate
  - → Pressure recovery with CFD model
  - → Same trend in both models.
    - Increases towards closed end

#### **Key result:**

 Average pressure loss in 1D model is irreversible (no pressure recovery)
 → more conservative







# CFD visualization: pressure recovery

Velocity (a) Instantaneous velocity field 0.000e+00 1.5 3 4.5 5.000e+00 Pressure loss due Pressure drop due Pressure Recovery to turbulence to dynamic head (b) Average pressure reduction 0.0001-00 0.00 1.0 1.00 5.000-00 (c) Velocity rms 66501-459 6-016 L809 \$5000 \$.7779-459

# CFD animation with orifice





# CFD animation without orifice





#### **Conclusions**

- 1D pulsation model validated with field data: good agreement
- Predicted pressure losses consistent between 1-D and CFD models
- CFD simulation provided more information, incl recovered pressure loss
- Well-designed orifice plates and other pressure drop elements:
  - → Are beneficial for reducing pulsations, vibrations and cavitation risks
  - → Can be used in the suction piping of reciprocating pumps if properly sized for mean and dynamic pressure losses
- Orifices not yet implemented more validation needed (ongoing)

