



49TH TURBOMACHINERY & 36TH PUMP SYMPOSIA

DECEMBER 7-10, 2020 | NOW VIRTUAL



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TURBOMACHINERY LABORATORY
TEXAS A&M ENGINEERING EXPERIMENT STATION

Optimizing Compressor Packing Designs by Predicting Rod Temperatures

Andreas Brandl, John Ladd, Karl Markey



HOERBIGER

Author Bios

Andreas Brandl is the Engineering Manager at HOERBIGER Service Inc. in Houston TX. His work focuses on Reciprocating Compressors for the Oil & Gas and Chemical/Petrochemical industry.

Before coming to Texas he was working in the corporate R&D department for HOERBIGER in Austria. Andreas earned his Master's degree in mechanical engineering at the Vienna University of Technology and his MBA at the Jones Graduate School of Business at Rice University.

John Ladd is a Solutions REE Engineer and Compressor Analyst at HOERBIGER Service In. in Houston, TX. His primary role is conducting comprehensive technical evaluations of reciprocating compressors to identify and quantify unit improvements in reliability, efficiency, and environmental soundness. Prior to his current role John earned his Master's in mechanical engineering at Colorado State University with a focus on legacy integral pipeline compressors.

Karl Markey is the R&P Engineering Team Manager at HOERBIGER in North America. His team focuses on product design and support for compressor OEM's and end users in a variety of industries. Karl has 15 years of experience in the Rings and Packing business and he has additional prior experience with product design and development outside the oil and gas industry. Karl has a Bachelor of Science degree from Massachusetts Institute of Technology and a Master of Engineering Degree from Rensselaer Polytechnic Institute.

Abstract

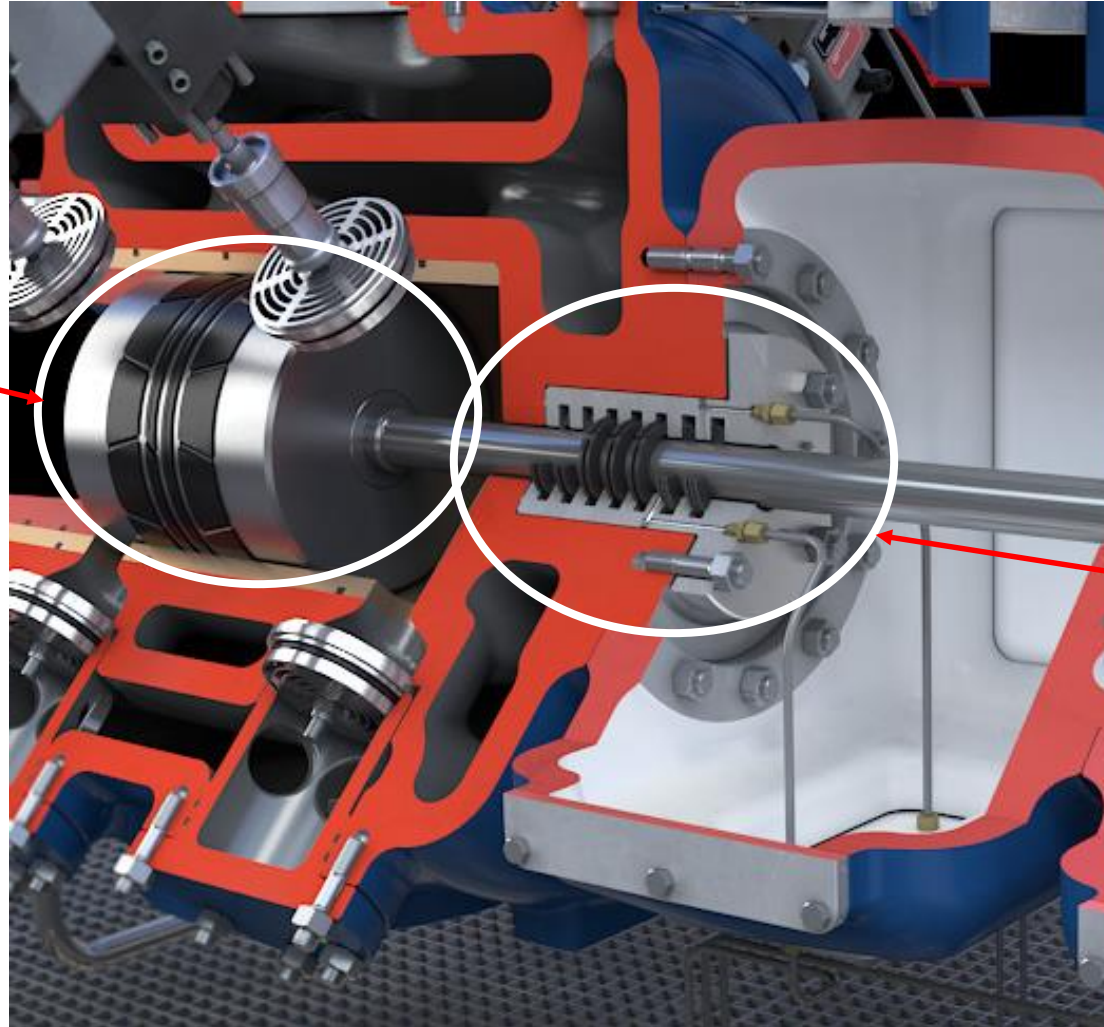
Current compressor rod packing engineering standards rely on rule of thumb methods and selection tables. This lecture presents a method to maximize the packing run time by optimizing the performance determining parameters of the packing. By predicting the rod temperature distribution, the performance of different packing layouts (case length, number of rings, backup ring material, ring design, case ID) can be compared and a relative run time prediction can be given. The lecture describes the model details, discusses the heat transfer correlation assumptions and deduces general packing design guidelines based on the model results. Several packing optimizations have been performed using this approach and two case studies are discussed in this lecture.

Agenda

- 1 Introduction and Motivation**
- 2 Description of the Rod Temperature Model
- 3 Optimization examples
- 4 Summary

Seal System – Piston & Pressure Packing

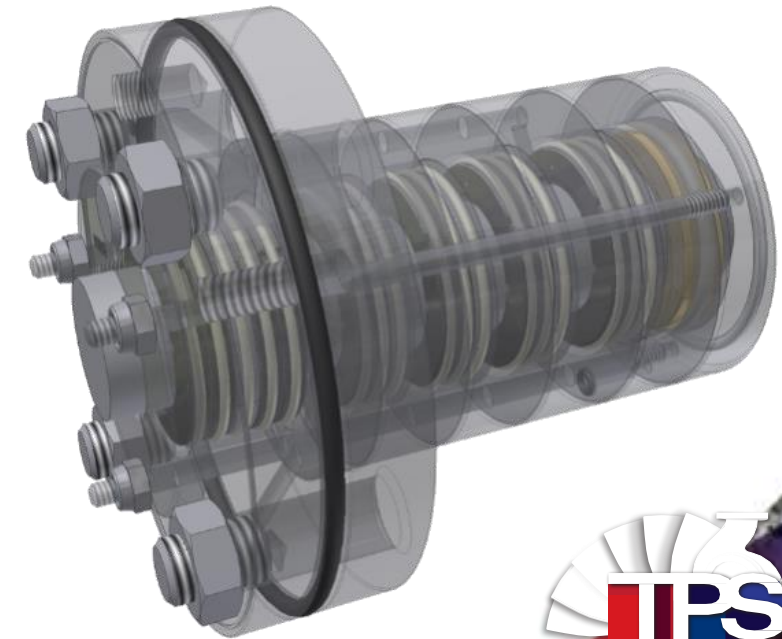
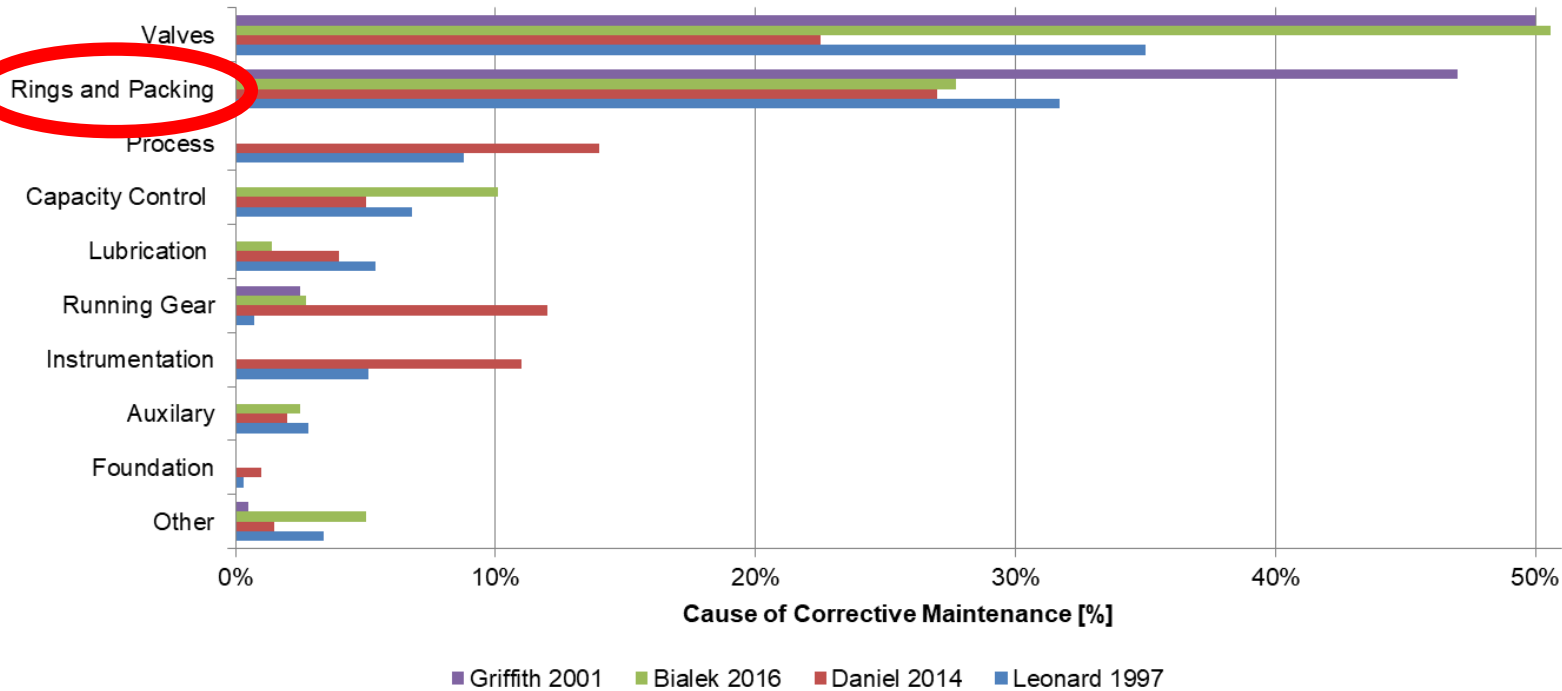
Compressor
Piston



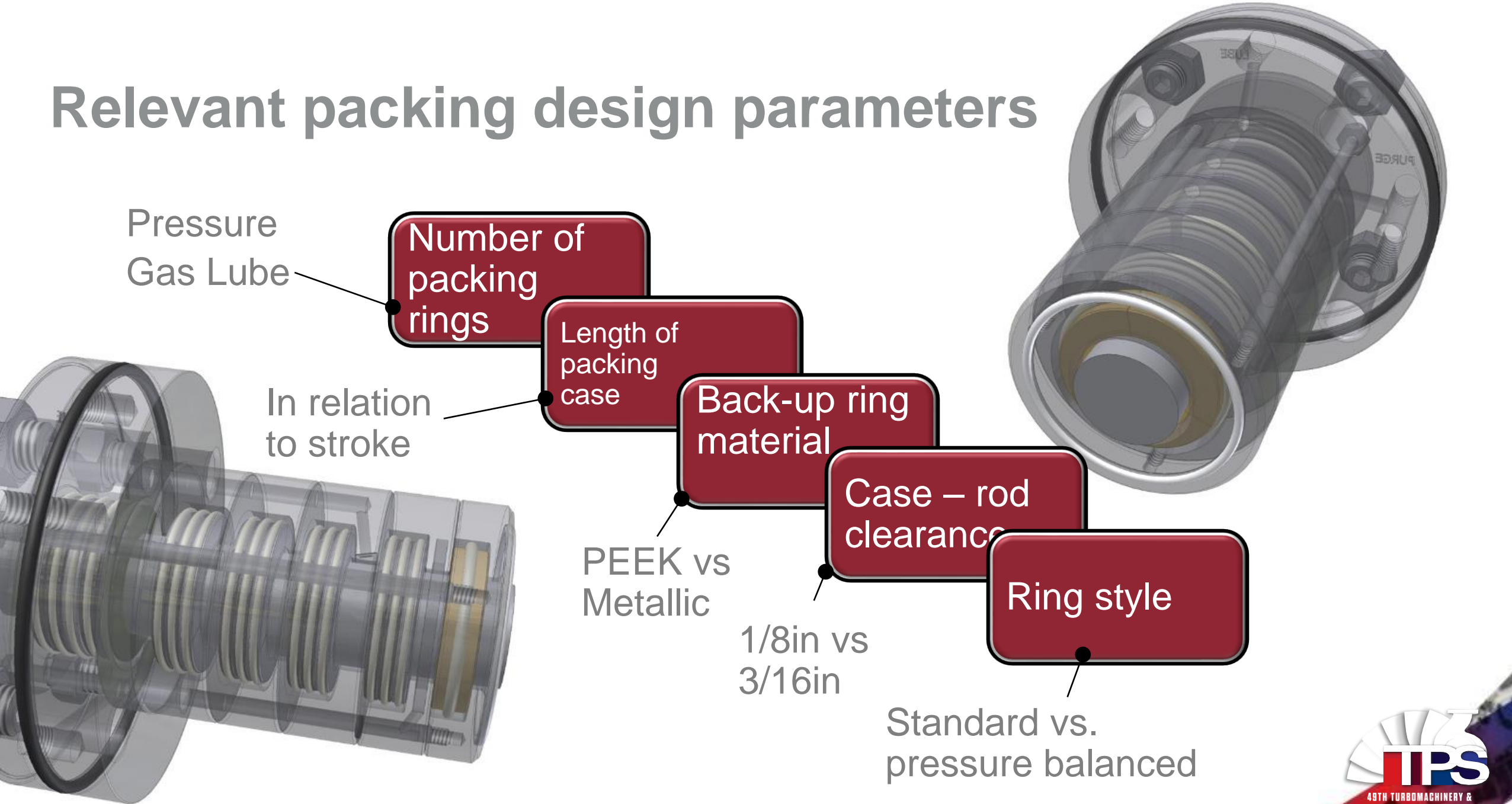
Pressure
packing

Seal system = Rings and Packing

27% - 47% of corrective maintenance events!

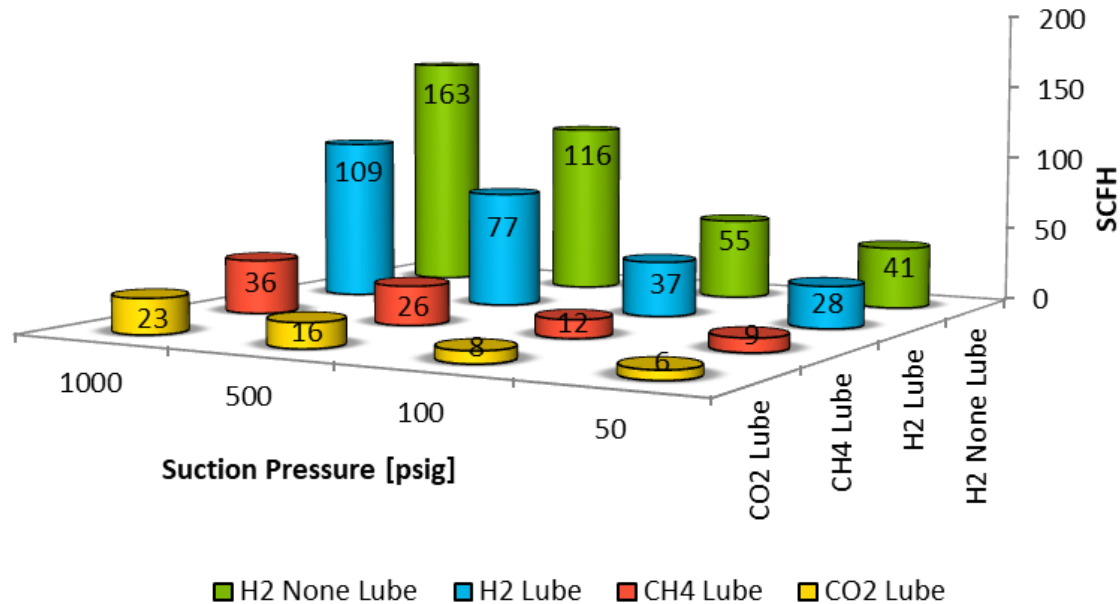


Relevant packing design parameters

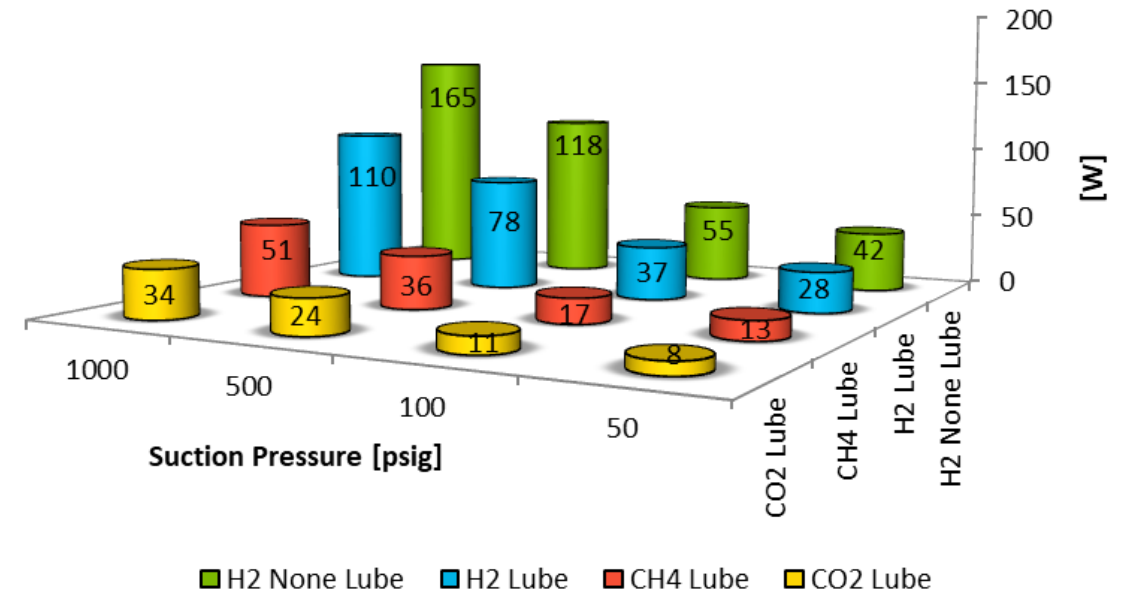


Typical leakage rates and heat removal

Expected Packing Leakage Rates (Based on empirical data)



Expected Heat Removal From Rod via Gas Leakage



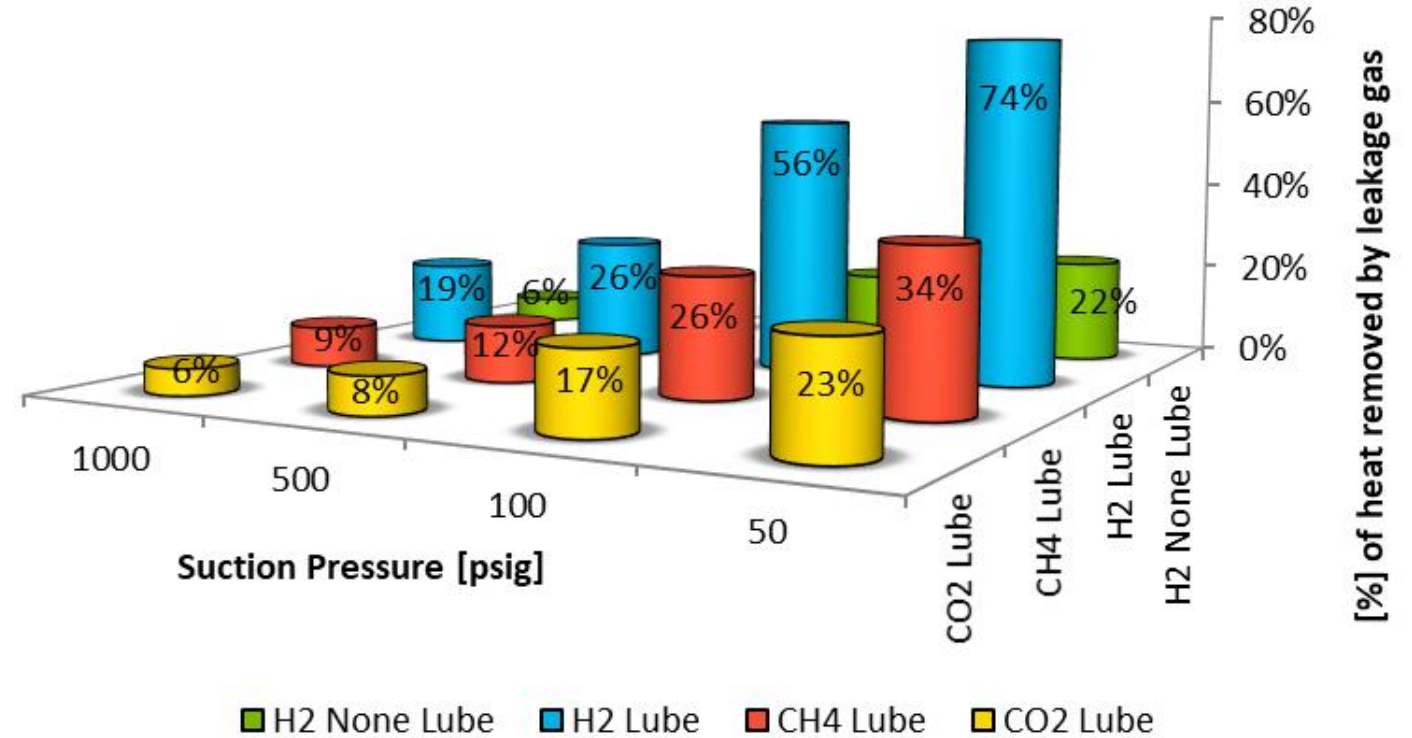
Assumptions: Compression ratio of 2.5, 2in rod dia, 9in stroke

Assumptions: Compression ratio of 2.5, 2in rod dia, 9in stroke, temperature rise of gas of 180°F

The leakage gas removes only a small fraction of the generated heat

Assumptions

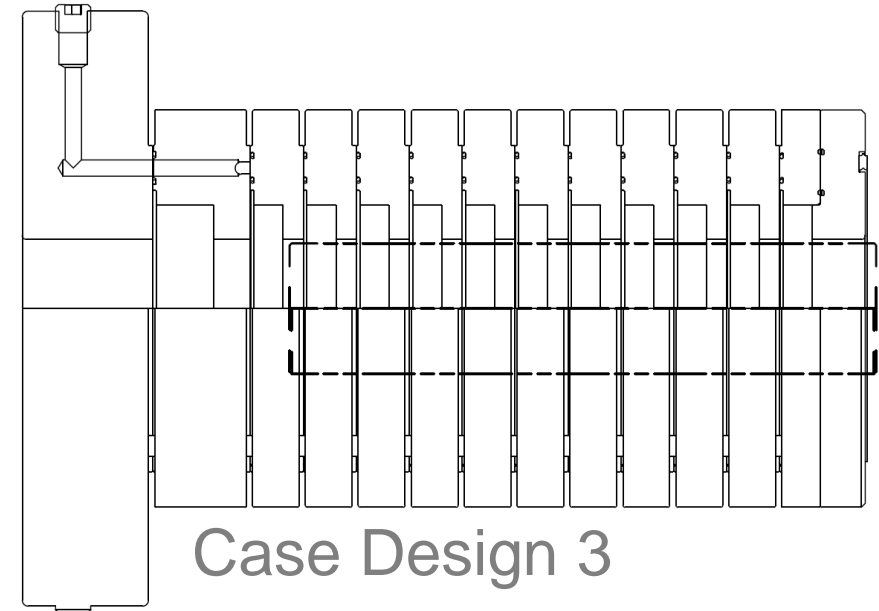
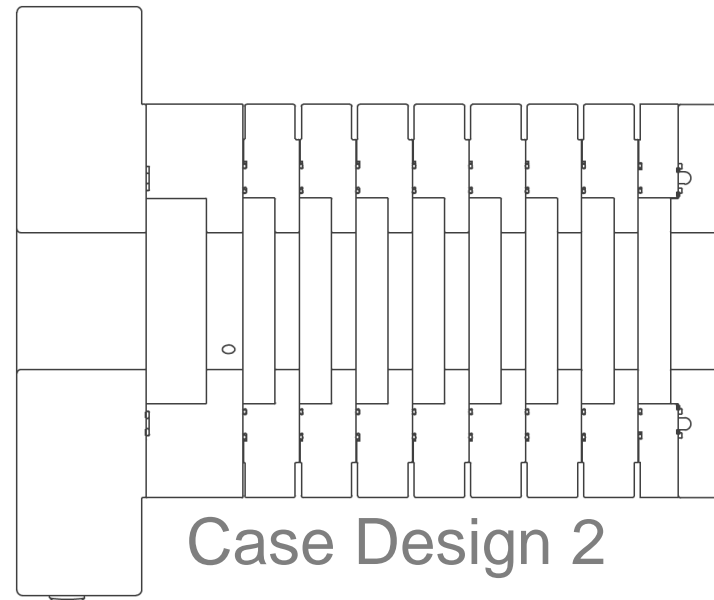
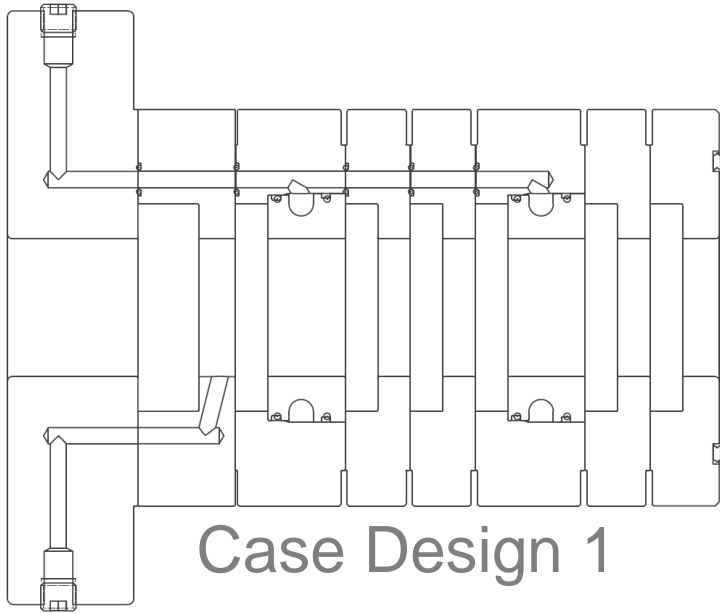
- Compression Ratio: 2.5
- Rod dia: 2in
- Stroke: 9in
- Temperature rise: 180°F
- Frictional coefficient:
 - 0.2 (none-lube)
 - 0.04 (lube)



Lube oil

Heat removal from packing lube oil is negligible

Ideal Packing Case – Rod Temperature is a key parameter



	Case Design 1	Case design 2	Case design 3
Number of rings	6	8	11
Case Length [in]	9	9	11
Rod Temp [°F]	?	?	?

Agenda

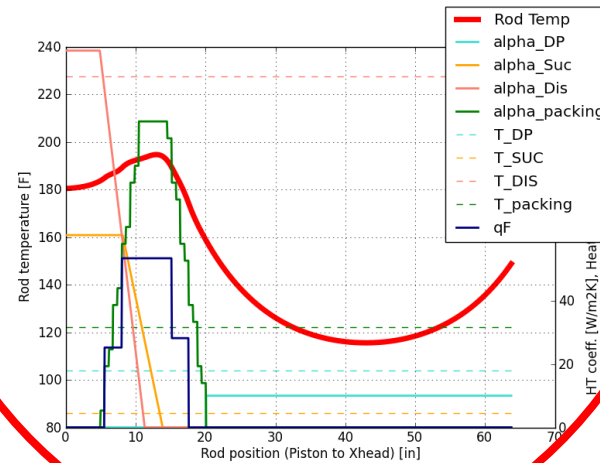
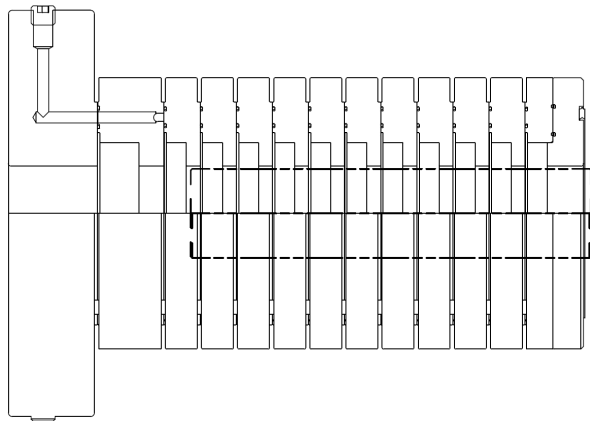
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Optimization Process – Increase the ring count without increasing the rod temperature

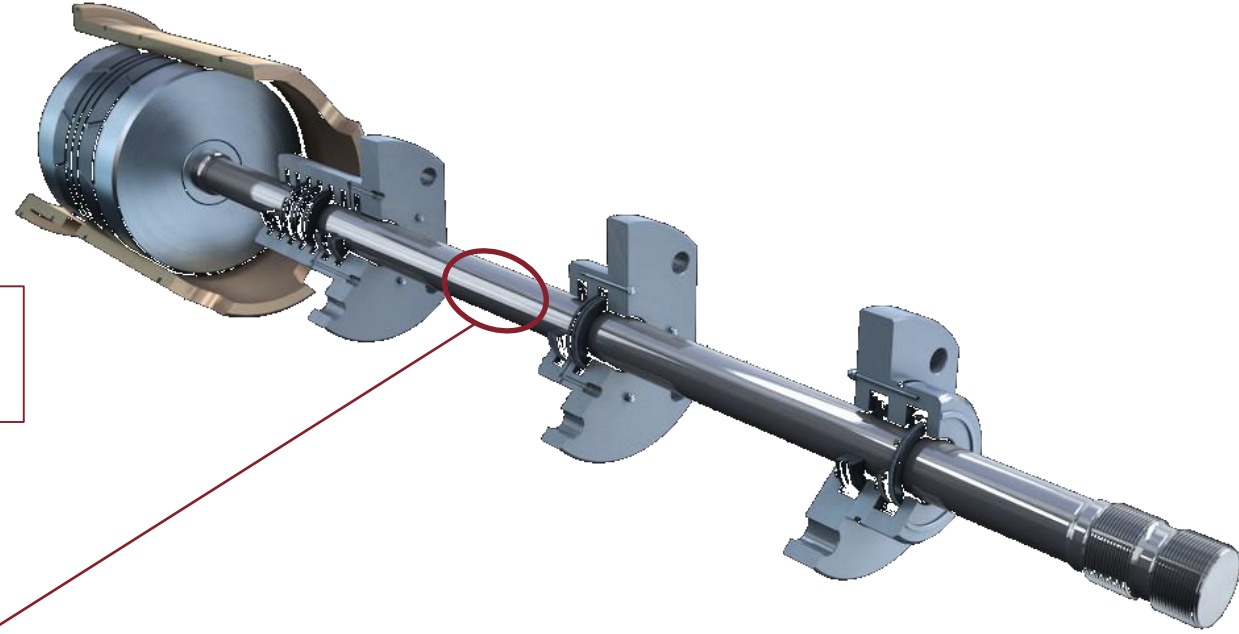
Add packing rings, increase the case length

Verify that the rod temperature does not increase

Increased packing run time



Thermal Model – Heat Generation, Heat Conduction, Heat Transfer



Heat transfer to the surrounding

Frictional heat generation

$$(\alpha(T_a - T_j))d\pi dx$$

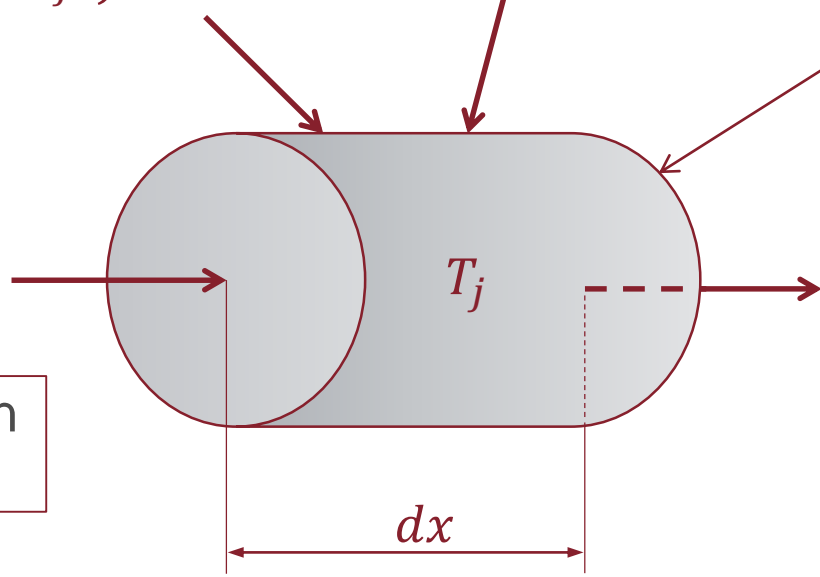
$$\dot{q}_{fr}d\pi dx$$

$$\dot{q}_x \frac{d^2\pi}{4}$$

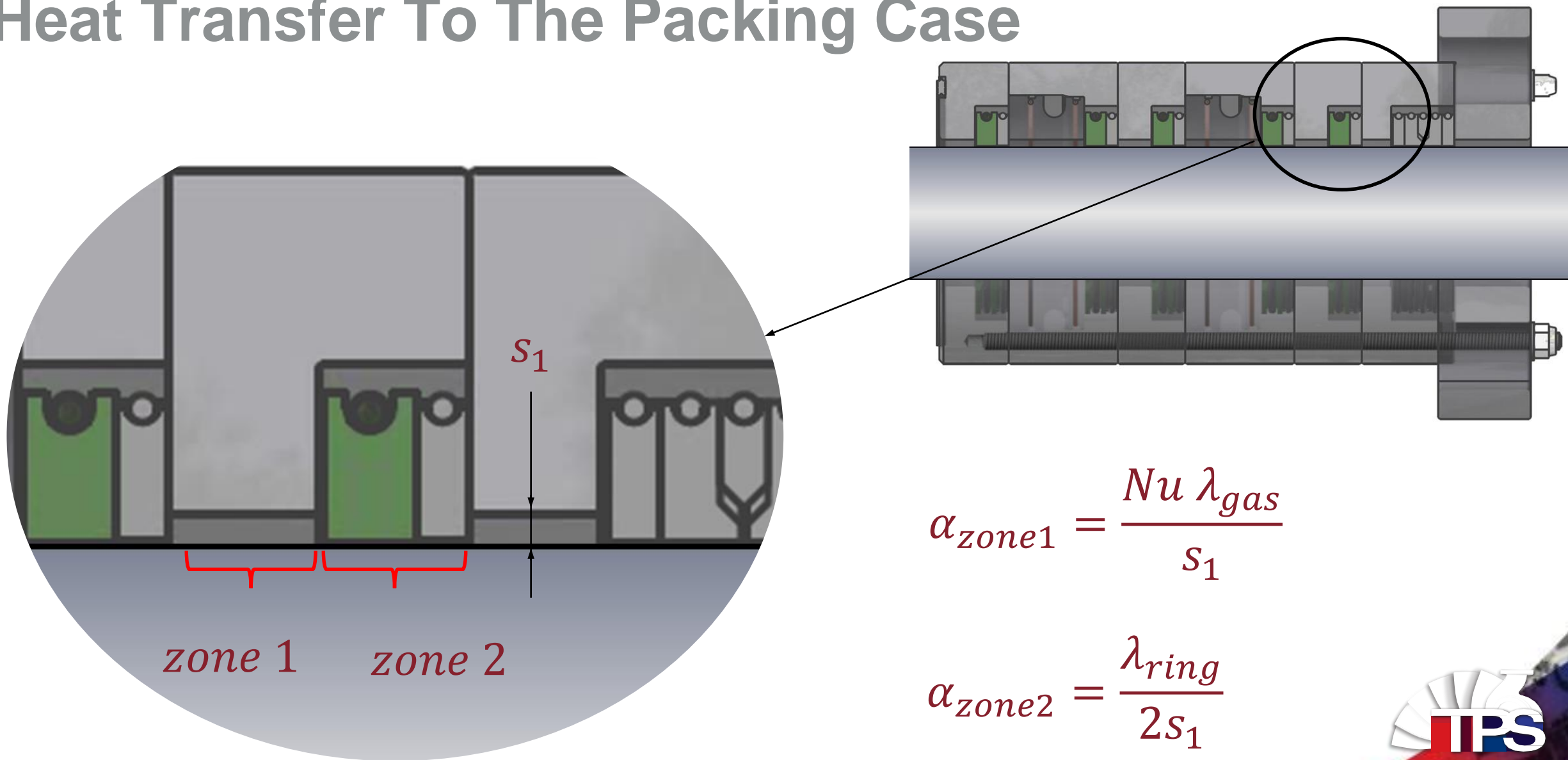
$$\left(\dot{q}_x + \frac{d\dot{q}_x}{dx} dx \right) \frac{d^2\pi}{4}$$

Heat conduction along the rod

Heat conduction along the rod



Heat Transfer To The Packing Case



$$\alpha_{zone1} = \frac{Nu \lambda_{gas}}{s_1}$$

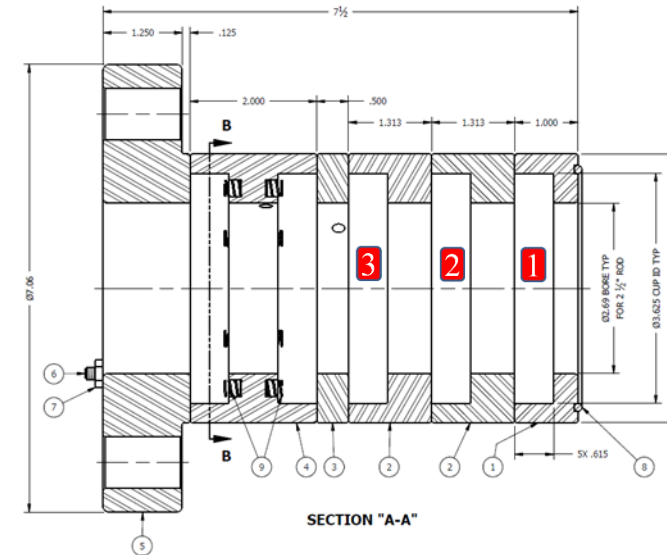
$$\alpha_{zone2} = \frac{\lambda_{ring}}{2s_1}$$

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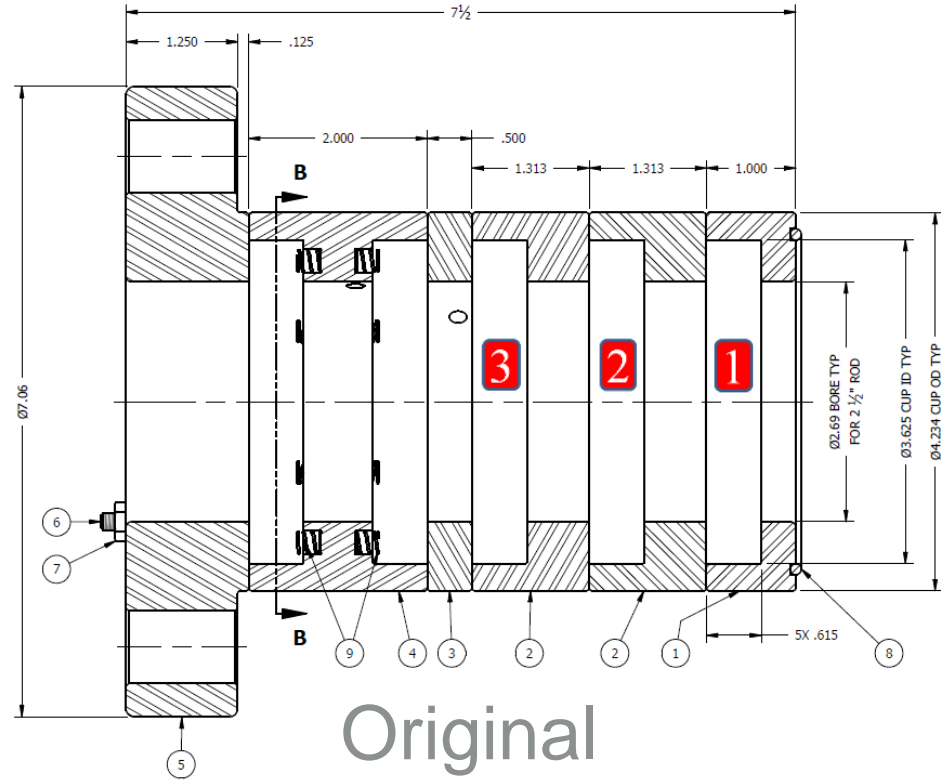
Low Pressure, Ethylene, None Lube

Parameter	
Speed [rpm]	440
Cylinder dia	16.5
Stroke [in]	9
Avg. p. speed [ft/min]	660
Rod dia [in]	2.5
Isentropic Exp. []	1.29
Molar mass [kg/kmol]	25.8
Cylinder Lubrication	None Lube
p_{suc} [psig]	14
p_{dis} [psig]	65

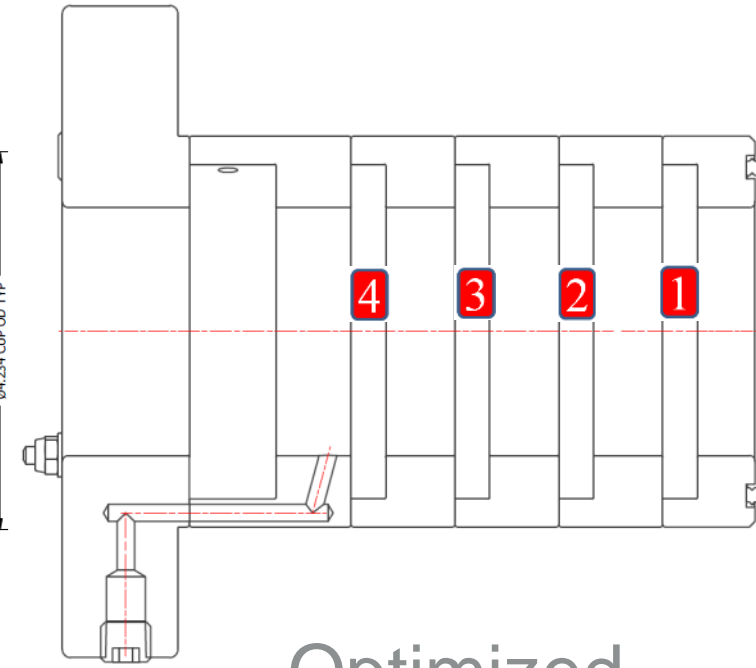


Packing Parameter	
# of seal rings	3
Length of case [in]	6.25
Back up ring material	NA
Rod – case cl. [in]	3/16
Ring style	RT

Packing Optimization



Original



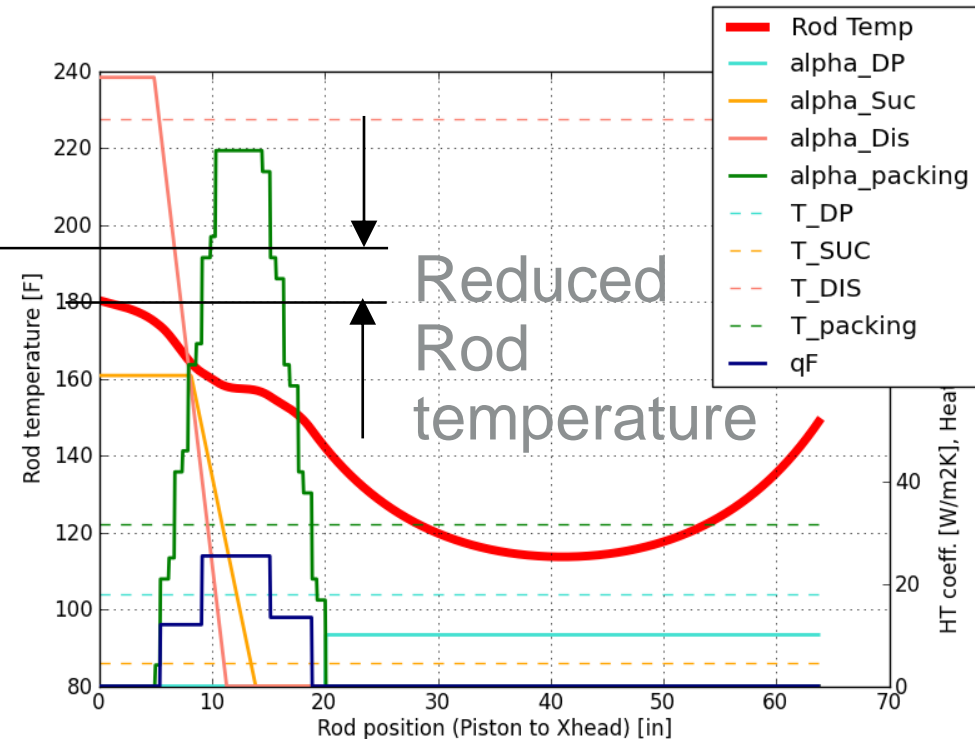
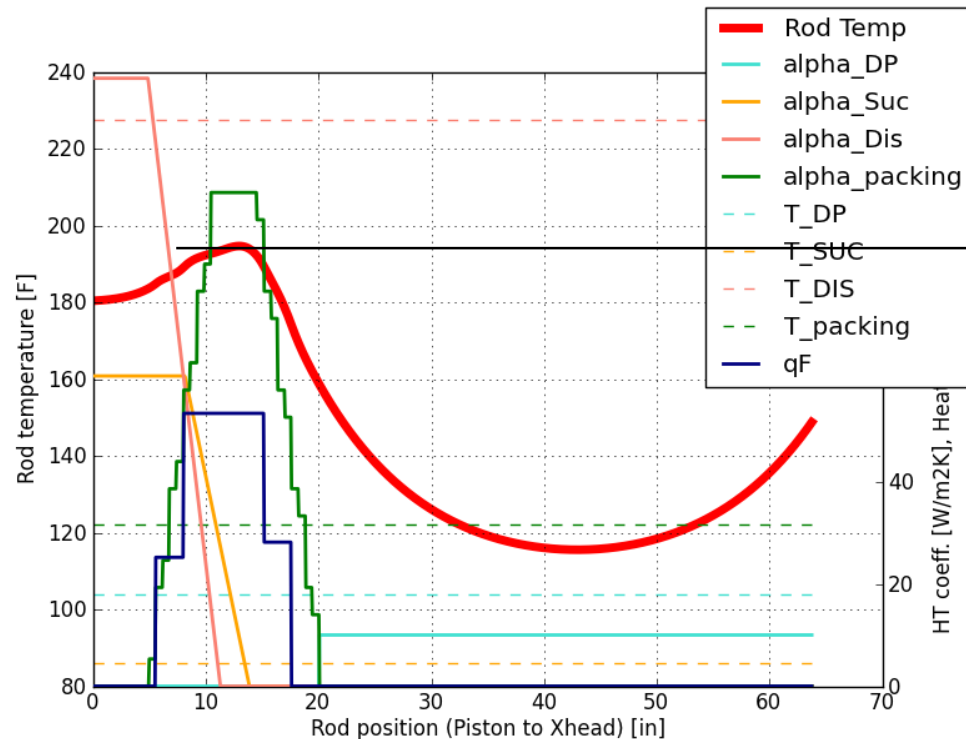
Optimized

- 1) Radial / Tangential cut ring
- 2) Pressure balanced ring

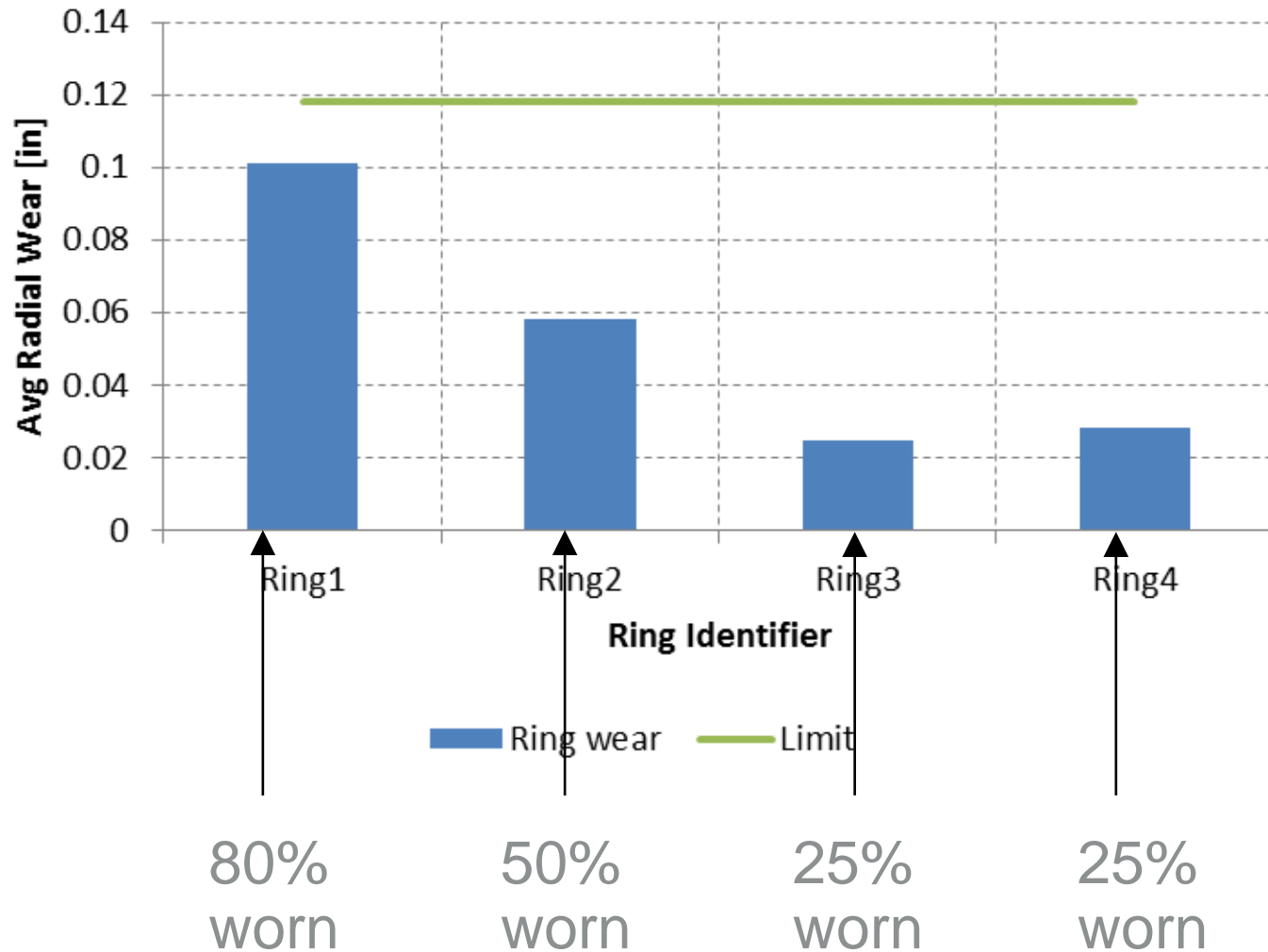
Packing Parameter	Existing	New	Impact
# of seal rings	3	4	Increased wear life
Length of case [in]	6.25	6.25	NA
Back up ring material	NA	NA	NA
Rod – case clearance [in]	3/16	3/16	NA
Ring style	RT ¹⁾	PB ²⁾	Reduced heat generation

Increased number of ring + Reduced Rod Temperature

- Increased number of rings
- Pressure balanced packing rings

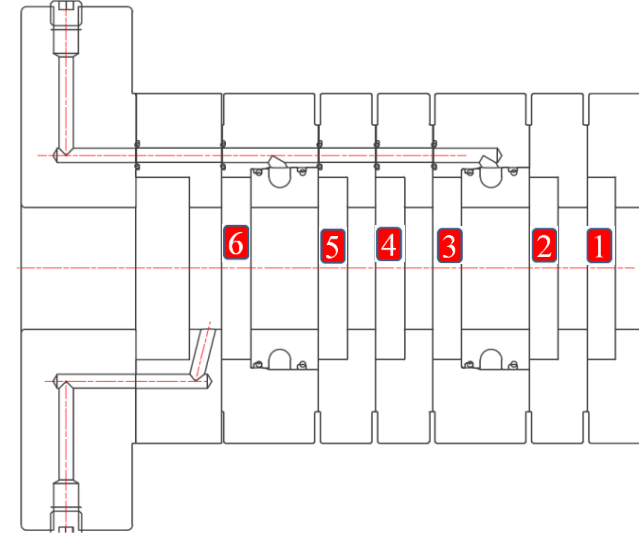


50% wear progress after 14 months of run time



Medium Pressure, Hydrogen, None Lube

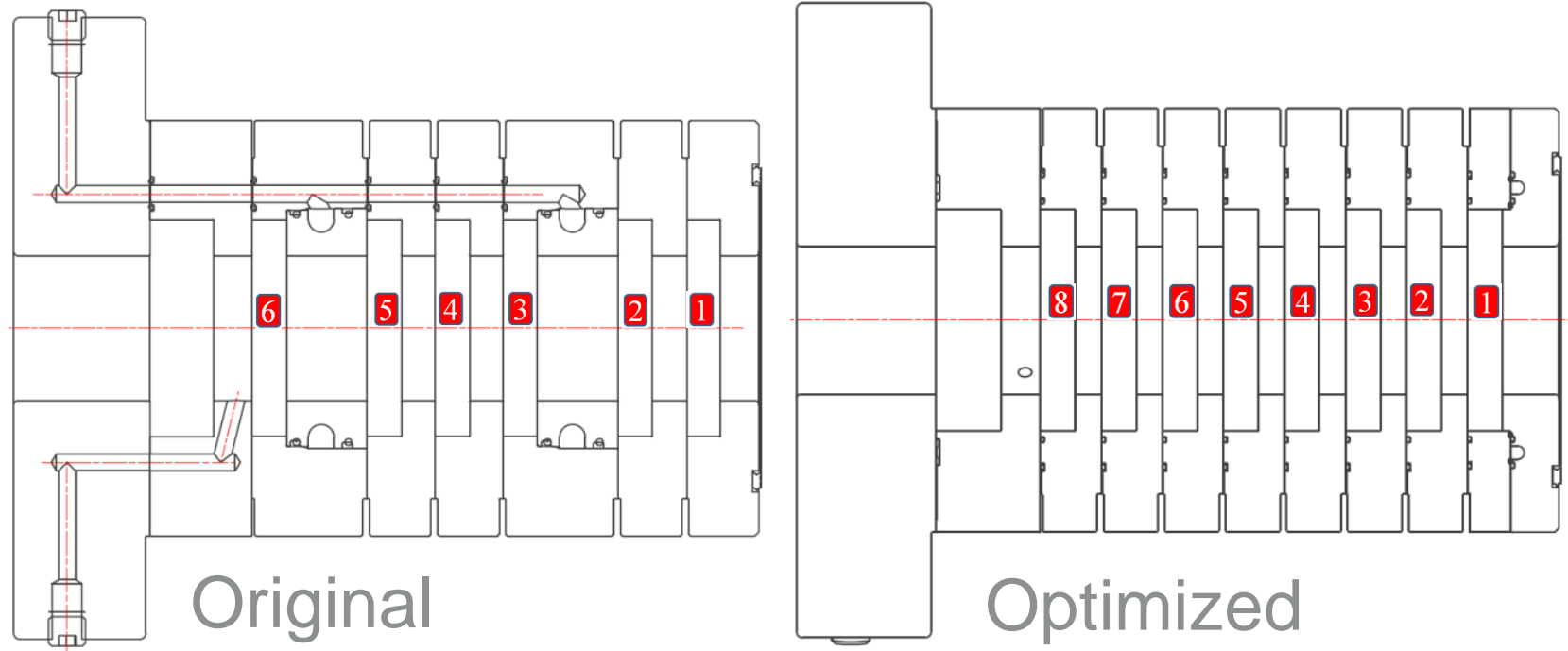
Parameter	
Speed [rpm]	396
Cylinder dia [in]	4.75
Stroke [in]	9
Avg. P. speed [ft/min]	593
Rod dia [in]	2
Isentropic Exp. []	1.4
Molar mass [kg/kmol]	2
Cylinder Lubrication	None Lube
p_{suc} [psig]	920
p_{dis} [psig]	1530



Packing Parameter	
# of seal rings	6
Length of case [in]	9
Back up ring material	PEEK
Rod – case cl.[in]	1/8
Ring style	RT

Packing Optimization

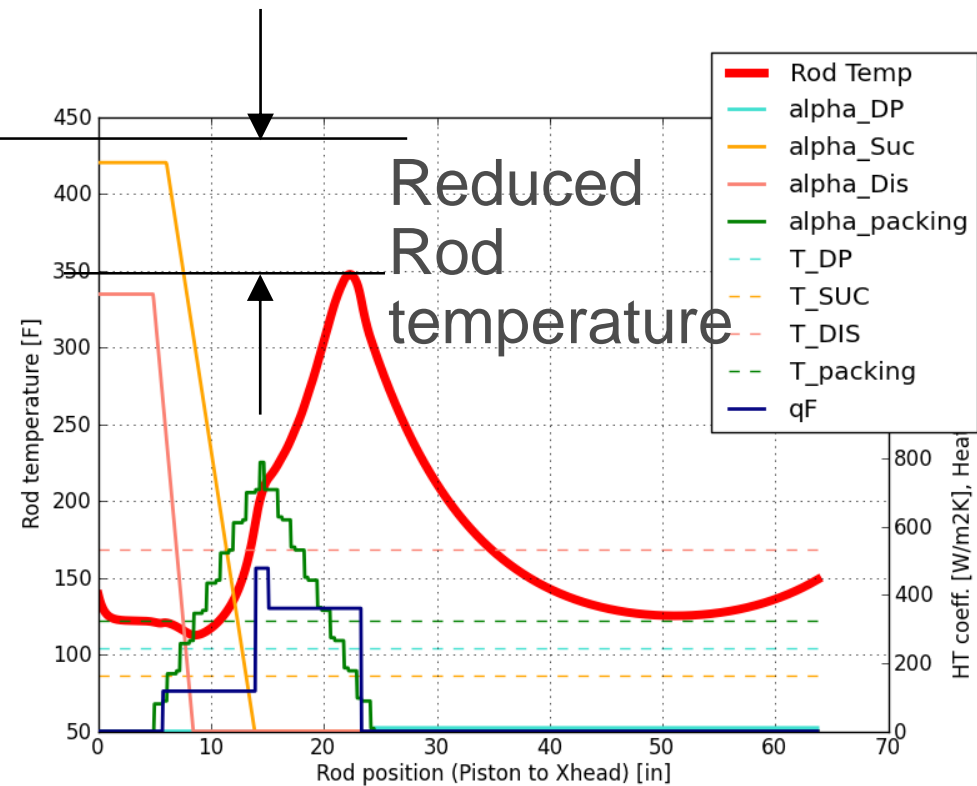
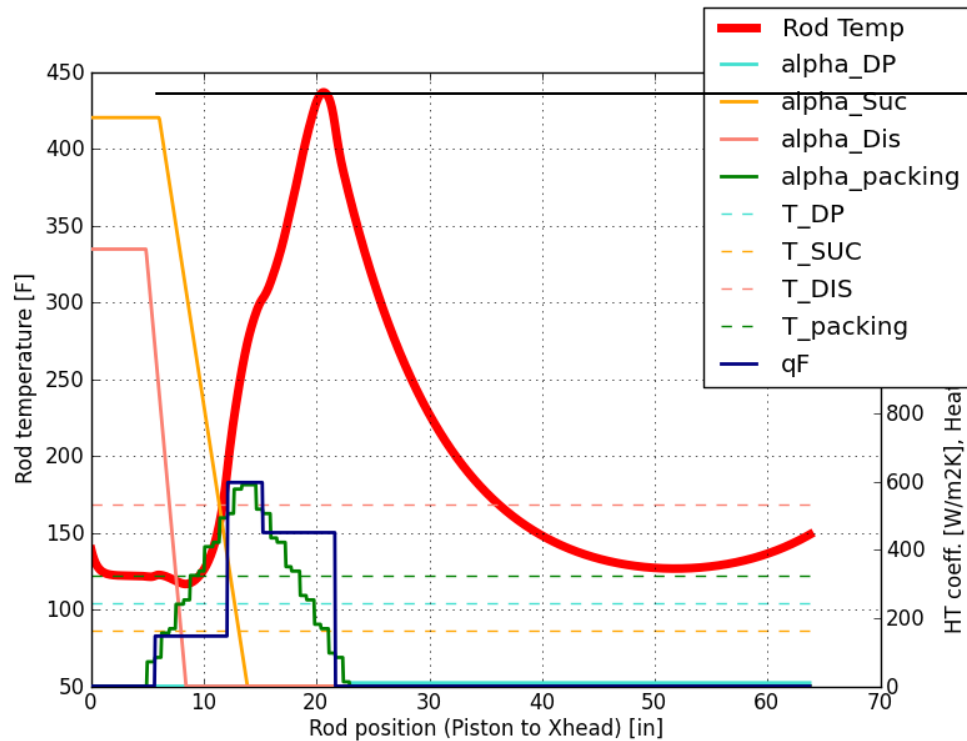
- 1) Radial / Tangential cut ring
- 2) Pressure balanced ring



Packing Parameter	Existing	New	Impact
# of seal rings	6	8	Increased wear life
Length of case [in]	9	9	NA
Back up ring material	PEEK	Bronze	Better heat transfer
Rod – case clearance [in]	1/8	1/8	NA
Ring style	RT ¹⁾	PB ²⁾	Reduced heat generation

Rod Temperature Distribution

- Increased number of rings
- Pressure balanced packing rings
- Metallic back-up ring



Agenda

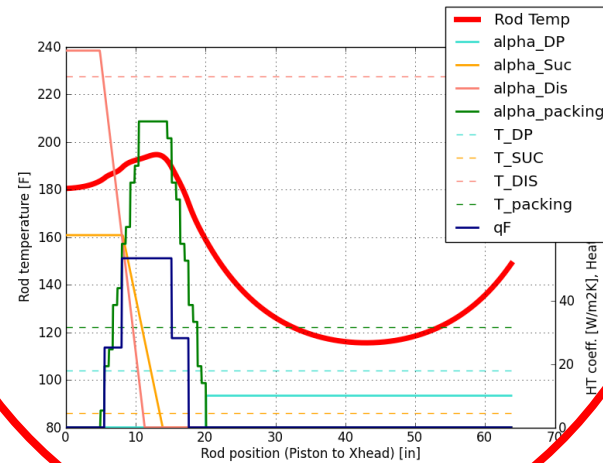
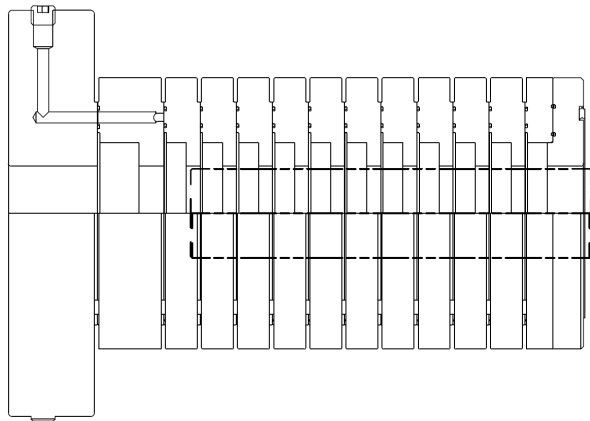
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Optimization Process – Increase the ring count without increasing the rod temperature

Add packing rings, increase the case length

Verify that the rod temperature does not increase

Increased packing run time



Making Pressure Packings more Reliable by Taking the Rod Temperature into Consideration

