

Troubleshooting of
Start-up for
Rerated & Old Steam
Turbine

A MITSUBISHI HEAVY INDUSTRIES COMPRESSOR

#### Presenter/Author Bios



#### Ayush Singh ayush.singh.p3@mhi.com

Engineer, Turbine Engineering & Design Section, Mitsubishi Heavy Industries Compressor Corporation, Hiroshima, Japan. He has 1 year of experience as a turbine engineer.

Mr. Ayush graduated from Indian Institute of Technology Kanpur, India.



#### Tomoaki Nogami tomoaki.nogami.r8@mhi.com

Engineer, Turbine Engineering & Design Section, Mitsubishi Heavy Industries Compressor Corporation, Hiroshima, Japan. He has 7 years of experience as a turbine engineer. Mr. Nogami graduated from The University of Tokyo.



#### Yusuke Oishi yusuke.oishi.er@mhi.com

Team Leader, Turbine Engineering & Design Section, Mitsubishi Heavy Industries Compressor Corporation, Hiroshima, Japan. He has 12 years of experience as a turbine engineer. Mr. Oishi graduated from Tokuyama College of Technology (Mechanical & Electrical Engineering).



#### Takuro Koda takuro.koda.k2@mhi.com

Manager, Turbine Engineering & Design Section, Mitsubishi Heavy Industries Compressor Corporation, Hiroshima, Japan. He has 15 years of experience as a turbine engineer. Mr. Koda graduated from Kyoto Institute of Technology.

#### Problem Statement

During the start-up of an old steam turbine (about 30 years old), that was internally revamped, it was tripped due to high rotor vibrations.

Site investigation was done to find out the root cause. Abnormal casing deformation and rubbing between labyrinth and rotor was found during the investigation.

In this study we would like to introduce the detail phenomenon, the root cause analysis, what we learned and what we should have done. We hope this would be beneficial for EPC and the end user to know the impact in case the old steam turbine was partially revamped.

#### Contents

- 1. Background
- 2. Site Observations
- 3. Root Cause Analysis (RCA)
- 4. RCA Summary
- 5. Site Countermeasure
- 6. Lessons Learned

During start-up of old turbines, approximately 30 years old,

- Steam turbine was tripped due to high vibrations.
- Multiple start-up trials were unsuccessful due to high rotor vibrations.
- Due to this reason, the turbine start-up was problematic.

The turbine spec for this case study is given in Table-1.

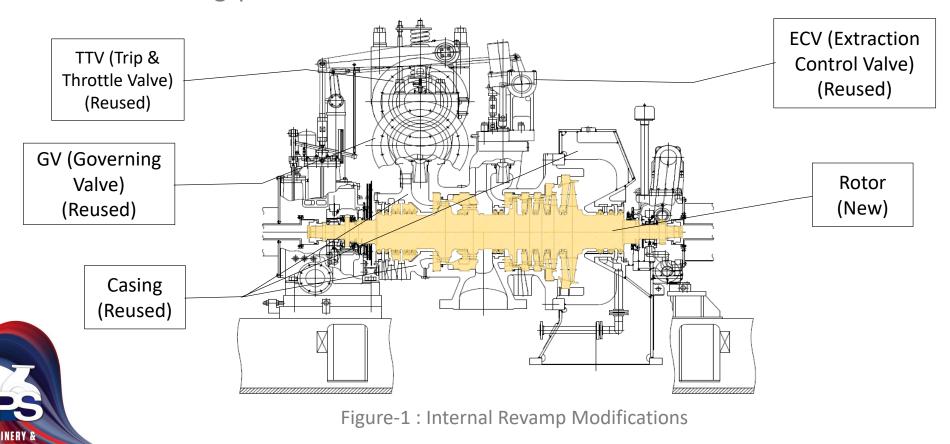
Table-1 : Turbine Spec				
Service	Synthesis Gas			
Model	Extraction Condensing Turbine			
Max. Output	Approx. 28,000 kW			
Max. Speed	Approx. 12,000 rpm			
Inlet Steam	106 ata x 505 deg-C			

• Operation history for the turbine is described in Table-2

Table-2: Operation History

Year	1986/3	~	2020/9	2020/9,10
Action	Shipment of original turbine	Continuous Operation	Internal revamp (To decrease the steam consumption and increase the performance)	Site operation after internal revamp
New Parts	Steam turbine assembly	-	Rotor, blade, nozzle, diaphragm, labyrinth seal	-
Design Philosophy	Old	$\rightarrow$	Latest	$\rightarrow$
Note	More than 30 years of operation. Sometimes during the past solo run tests abnormal speed increase around 2 <sup>nd</sup> warming up speed was observed. But no signification issues like high vibrations were observed.		[Reused parts] Casing, governing valve (GV), extraction control valve (ECV), trip & throttle valve (TTV) etc.  Combination of old design style ( reused casing without casing drain line) and new design philosophy (internal parts)	High vibrations were observed during start-up in multiple trials.

• In Fig-1, the highlighted parts (yellow) are revamped, and the remaining parts are reused.



Start-Up Curve

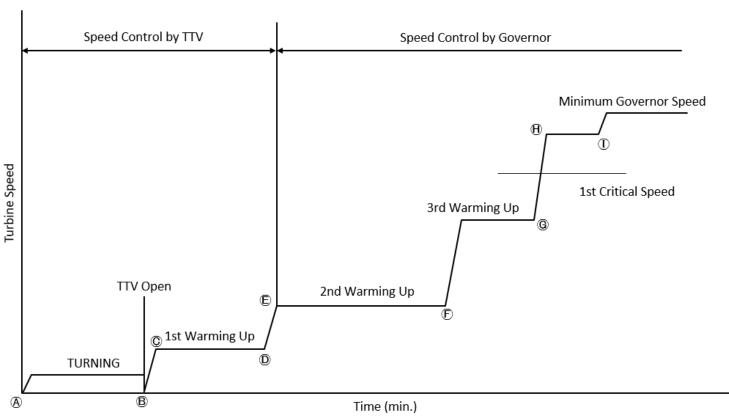


Figure-2: Start-Up Diagram

Remarks:

Upto the 1<sup>st</sup> warming-up speed, TTV speed control is applied, so that better uniform heating of turbine casing can be achieved.

## 2. Site Observations

Trials 1-4: During Commissioning Solo Run with Working Rotor

- Multiple trials were conducted by TTV start-up (speed increase by manual operation).
- In the 1<sup>st</sup> trial (Fig-3) there was a sudden increase in speed and vibration (Speed increase upto 4 times the target speed).
- After every trial, residual unbalance record showed varying rotor deformation.

From the 5<sup>th</sup> trial spare rotor was used instead of the working rotor (used for 1<sup>st</sup> – 4<sup>th</sup> trial).

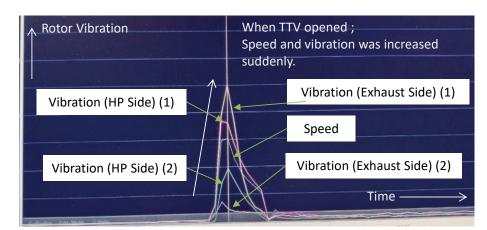


Figure-3: Speed & Vibration Trend for 1st Trial

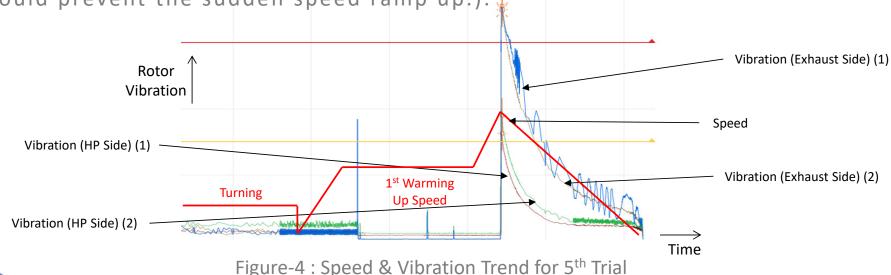
Remarks : Similar trend was observed for next 3 trials

Quality of manual start-ups depends on the operator skill. Therefore, this phenomenon (rubbing due to the sudden speed ramp up) may be occurred in the past operation. It is required to turn (open direction) the TTV handle slowly to prevent the sudden speed ramp up. Operator has to turn (close direction) the handle of TTV immediately when the rotor start the rotation to prevent the sudden speed ramp up. The sudden speed ramp up is occurred due to the friction of bearing is reduced from the large static friction to the small dynamic friction. The great skill is required for these operator activities (slowly open and quick close).

## 2. Site Observations

Trial 5<sup>th</sup>: Solo Run with Spare Rotor

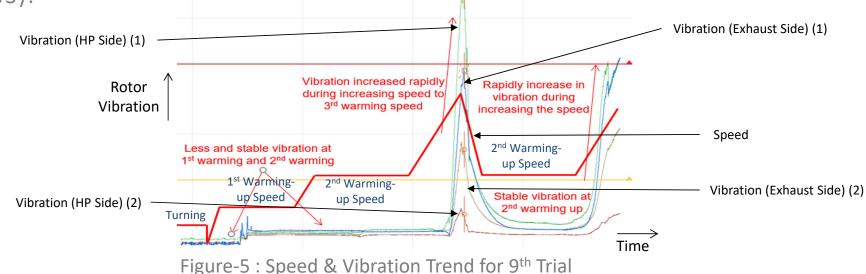
- Due to abnormal increase in rotating speed, there is a possibility of rotor rubbing with stationary parts.
- Labyrinth clearance was increased to prevent any potential rubbing of rotor.
- Similar to previous starts, speed control was difficult & turbine was tripped again due to high vibration (Fig-4).
- Due to the difficulty faced by operator in TTV start-up, GV start-up was applied from the next trial ( $6^{th}$  trial) and the result was a successful (This could prevent the sudden speed ramp up.).

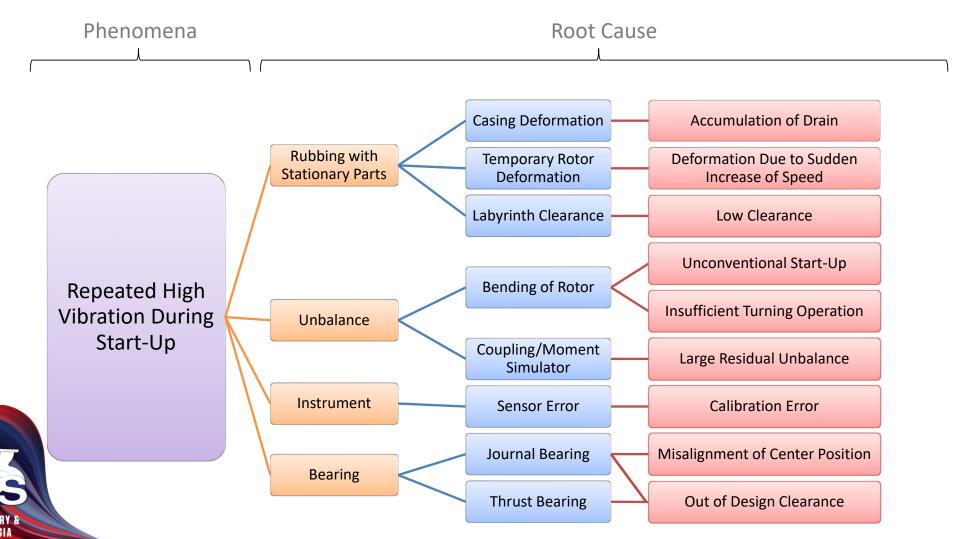


## 2. Site Observations

9<sup>th</sup> Trial: String Condition with Spare Rotor

- Speed control was a success & Speed was gradually increased (GV Start) to 2nd warming up speed without any issues (Fig-5).
- At 3rd warming up speed high vibrations occurred, so the speed was decreased.
- Vibration fluctuations were observed at different speeds.
- So, the turbine speed was increased and decreased based on the fluctuations and finally the operator was able to achieve Minimum Governor Speed (MGS).





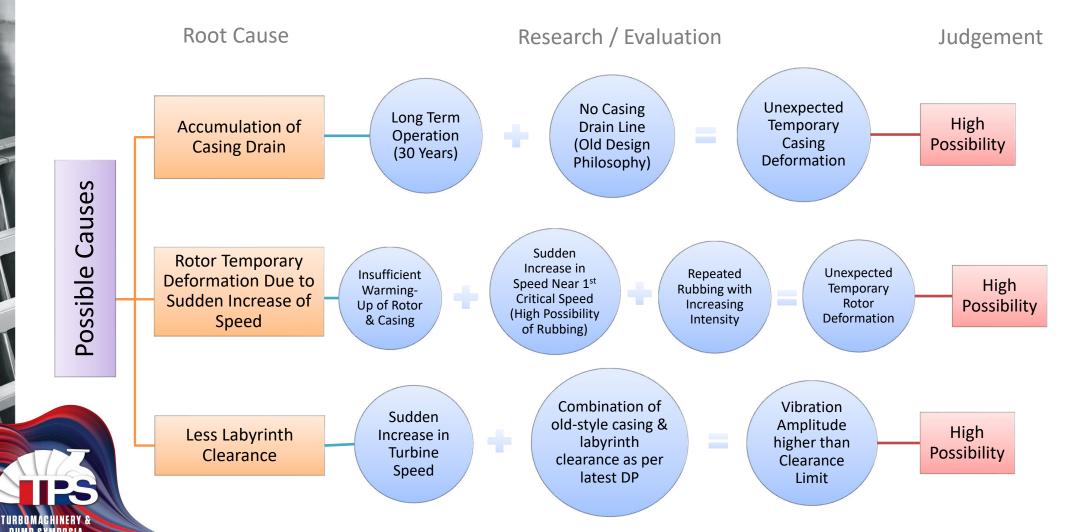
Root Cause How to Verify Verification Result

Check for drain High Accumulation of Drain Due to no drain line, accumulated drain was observed. Causes accumulation Possibility **Operation Record** Sudden increase in speed just after opening TTV. This might have caused High Deformation Due to high vibrations and rubbing with stationary parts. Sudden Increase of Speed Possibility Check Possible Operating clearance is lower than the current standard design. The operating clearance depends on the clearance at cold condition and the casing deformation. The cold clearance of this revamp project is the small clearance based on High Low Clearance Clearance Check the current standard design which consider the small casing deformation Possibility due to the casing drain removing. Casing deformation of this reused turbine is lager than the casing based on the current design which consider the casing drain removing. Therefore, operating clearance is lower than the current standard design.

Judgement

Root Cause How to Verify Verification Result Judgement

Unconventional Start-**Operation Record** Moderate Large increment in speed, which is not suitable for start-up. Check **Possibility** Causes up **Operation Record** Turning operation stopped in about 15 minutes. Due to this the lower half Moderate **Stopping Turning Operation** before Start-up of shaft may be heated up by sealing steam. Check Possibility Residual Unbalance Large Residual Possible Residual unbalance was within the criteria. Not Possible Record Check Unbalance Sensor Calibration Calibration Record Calibration record showed no problem. Not Possible Error Check Misalignment of Center **Operating Data** Shaft center position deviation was within criteria from our experience of Not Possible Check similar turbine. **Position** Out Design Clearance Clearance Check Bearing clearance was within the acceptance criteria at assembly stage. Not Possible



## 4. RCA Summary

After 1<sup>st</sup> Revamp

Drain Accumulation

Sudden
Increase in
Operating
Speed

High Vibrations

- Internal parts were revamped
- Existing Casing was reused
- Less clearance compared to original case (Latest Design Philosophy) with no drain line
- Gradual accumulation of condensate due to lack of casing drain resulted in thermal bending of casing
- Speed control by manual TTV start-up was difficult due to deformed casing of old design style & insufficient operator skills
- Sudden increase in speed and repeatedly increasing rubbing resulted in temporary rotor deformation
- Rubbing of rotor with stationary parts

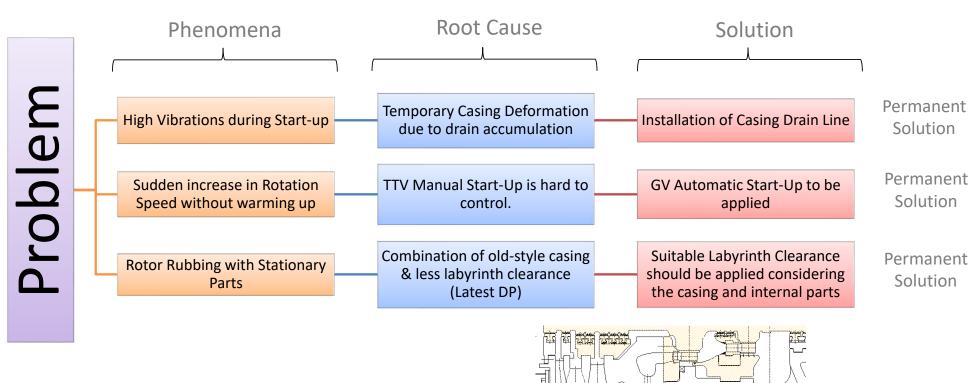


## 5. Site Countermeasures

Following countermeasures were taken for a robust start-up:

- TTV manual start-up was difficult due to deformed casing of old design style and insufficient operator skills.
- GV automatic start-up was applied instead of TTV manual operation. This allowed better control of the steam admission & therefore a slower speed ramp-up.
- If vibration is increased during transition, operating speed should be decreased to previous operating speed.
- Sudden change in operating conditions should be avoided except the critical speed band to perform the gradual warming-up.

#### 6. Lessons Learned



Added casing drain line is nearly low point.
Casing drain line is usually installed in the lowest point and this casing drain line is used the line for casing internal pressure measurement (PG, PT) as common. In this project, the existing casing internal pressure measurement (PT) line was used as the added casing drain line.

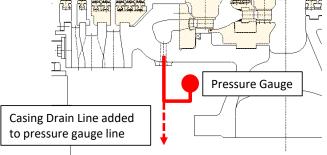


Figure-6: Casing Drain

### 6. Lessons Learned

Final conclusions from the case study:

- Casing drain line installation should be considered for the revamp of old steam turbines (Turbine age indicate if the casing drain line was installed or not).
- For turbines which are currently in operation and don't need revamp should also consider addition of casing drain line as a countermeasure for any future issues.
- Sudden change in operating conditions should be avoided to prevent any abnormal change in rotating speed.
- For deformed casing of old design style turbine start-up should be robust to prevent rotor deformation due to rubbing with stationary parts.
- If GV start-up is applied, large labyrinth clearance should be considered & customer's approval is required for performance deterioration.
- In case of internal revamp of old steam turbine, suitable labyrinth clearance should be applied.