Successful Avoidance of Major Secondary Damages Using Proximity Measurements to Detect a Partially Fractured Piston Rod

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Plant Location - Brunsbüttel, Germany





t / day	t / year
1,950	690,000
1,450	500,000
60	21,000
50	17,000
	1,950 1,450 60

Compressor Data

Machine type: 4-throw, horizontal, double-acting, 4-stage

Stroke: 300 mm (11.7 in.)

Piston diameter: 920, 500, 280, 160 mm (36, 20, 11, 6 in.)

Speed: 424 RPM

Power: 2,700 KW (3,600 HP)

Year of manufacture: 1977

Medium: CO2 (~98.5%)

Suction pressure: 1.8 bar (26 psig)

Discharge pressure: 165 bar (2,400 psig)

Flow rate: 13,000 Nm3/h

Capacity control: fine tuning by

suction pressure



GB-101B

Background

- Site has operated two identical 4-throw reciprocating compressors in carbon dioxide service since 1978.
- Since 2001, when an on-line condition monitoring system was installed on both compressors, numerous component failures have been detected by the system including: rider band wear, valve problems and loose components (valve cages, etc.)
- In 2002 the automatic machinery protection function of the system limited additional consequential damages caused by a piston rod failure, utilizing crosshead acceleration as the only shut down parameter.

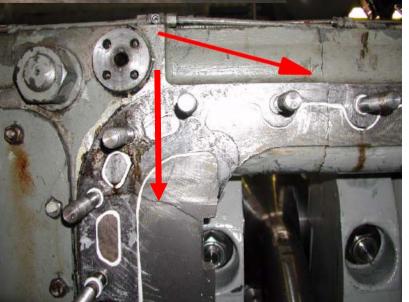
Failed Components - 2002 Rod Failure

Photos of damaged components

Damaged piston rod

Ripped

casing



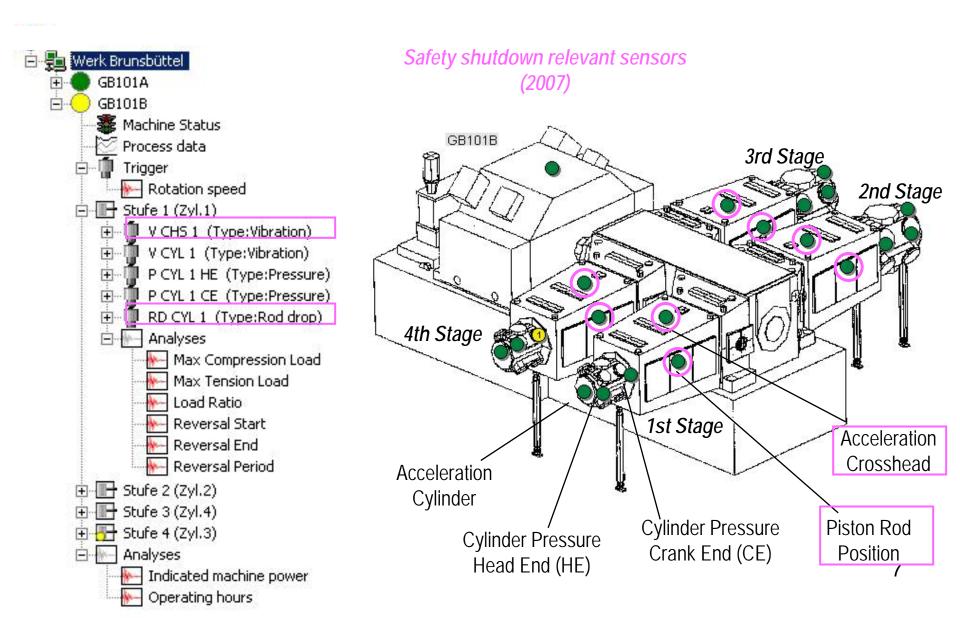
Bent con-rod

Background (cont'd)

• In 2006 piston rod position signals from the existing rod drop probes were added to the machinery protection function, initiated by YARA from the experience in 2002, that piston rod movement increases rapidly a short time prior to complete failure

 In Sept. 2007 the system successfully avoided potentially major cost-intensive damages to one compressor by detecting the development of a cracked 1st stage piston rod and automatically tripping the machine prior to complete failure, using peak-to-peak rod run out analyses.

Sensor Positions



Photograph of Affected Cylinder (1st stage)



Photograph of Proximity Probe (1st stage)



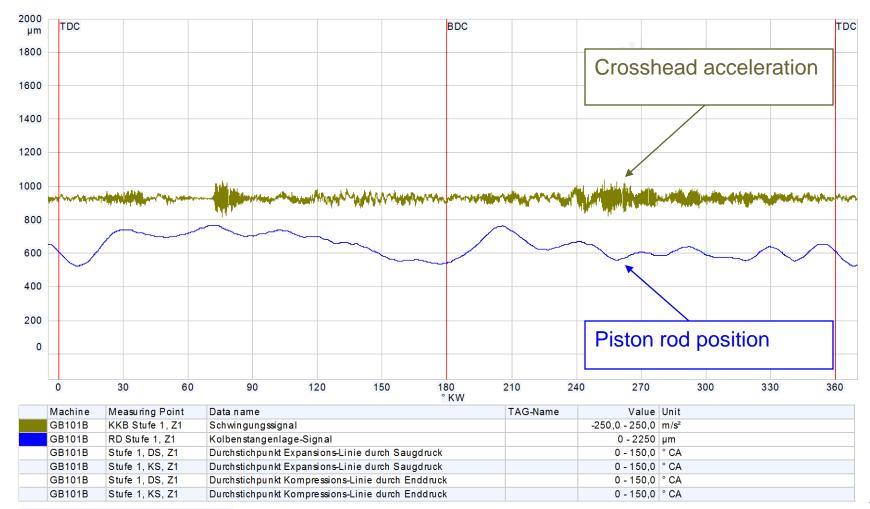
Sequence of Events Sept. 6, 2007

- Site planned to operate above rod load design limits to:
 - simulate a capacity increase of the urea process and
 - identify potential bottlenecks of the urea plant caused by an increased flow rate in advance of installation of an additional new compressor and HP equipment

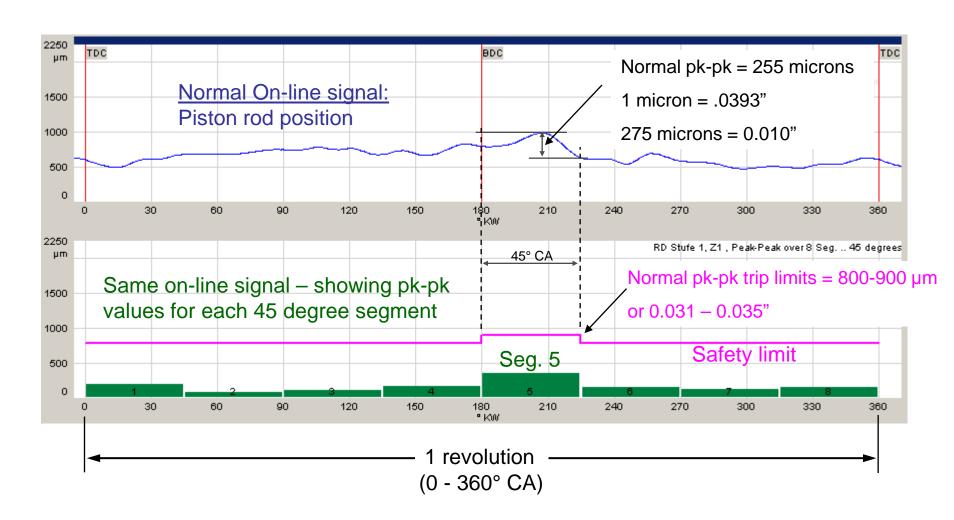
Note: After 6 years of experience with this machine monitoring system site was confident that the compressor would be safely tripped in time if the overloading caused mechanical problems.

 Trip limits for rod run out analysis were set corresponding to operating conditions

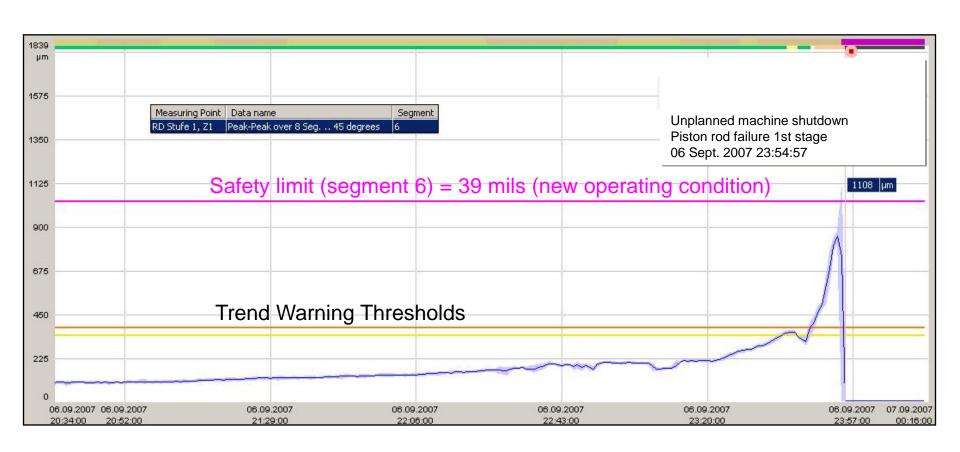
Data Analysis Crosshead Vibration & Rod Position Normal signatures 1st Stage - 10:00 am Sept. 6 (before change)



Definition of AnalysisPiston rod position, peak-to-peak, 8 segments



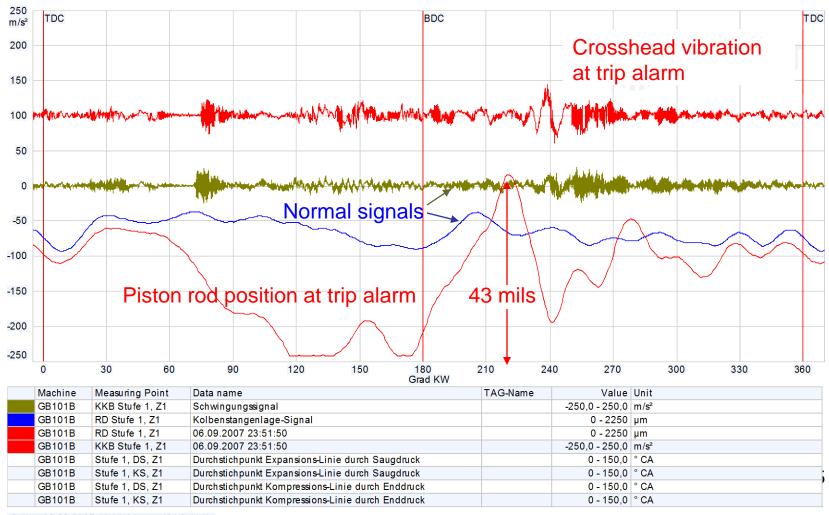
Data Analysis - 4 hr. Trend (following process change) Rod Position - Segment 6 (60-70 deg Crank Angle)



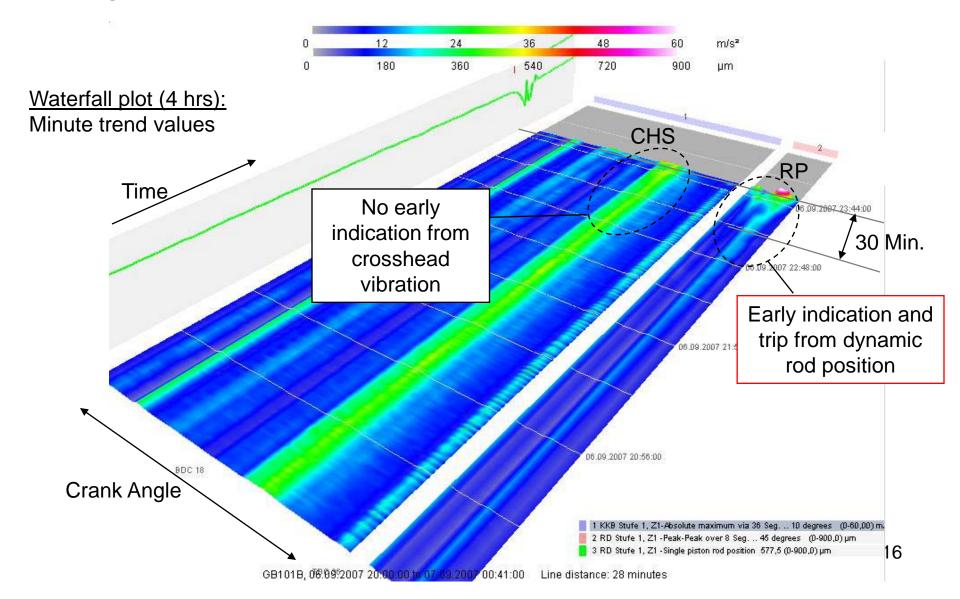
System Log BookWarning and Alarm Outputs (20 min.)

Туре	Date	Module	Measuring Point	5, 1	Message			
1 Information	06.09.2007 23:53:49	Analysis 1 Coo	Stop	!	Maschine steht			
A IMPORTANT	06.09.2007 23:53:44	Safety 5 Sec	RD Stufe 1, Z1 Trip	!	HzA: Roddrop-Schwingung oberhalb de	er Safety-Grenze		
A IMPORTANT	06.09.2007 23:53:43	Safety 2 Sec	RD Stufe 1, Z1	A 2	VzA: Roddrop-Schwingung oberhalb de	er Safety-Grenze		
A IMPORTANT	06.09.2007 23:53:42	Safety	RD Stufe 1, Z1		VzA: Roddrop-Schwingung oberhalb de	er Safety-Grenze		
A IMPORTANT	06.09.2007 23:53:41	Safety	RD Stufe 1, Z1	Ų.	Safety-Grenze in den folgenden Segme	enten überschritten; 6		
1nformation	06.09.2007 23:52:06	Pattern		A 1	Neues Zustands-Muster Nr. 25468 eing	getragen		
Event	06.09.2007 23:52:00	Alarm	KKB Stufe 1, Z1	+	<mark>9 24 Ab <mark>25 Ab</mark> 30 RM9 Absi<mark>17 Ab</mark> 36 F</mark>	RM17 Ab <mark>18 Ab 9 Abs 18 Ab 18 RM</mark> 24	RMFFT R 19 Ab20 Ab24 Ab20 RMF	FT R <mark>17 Ab</mark> 24 Al
Event	06.09.2007 23:51:00	Alarm	KKB Stufe 2, Z2	=1	FT RMS value			
Information	06.09.2007 23:49:02	Pattern			Neues Zustands-Muster Nr. 25467 eing	getragen		
Event	06.09.2007 23:46:57	Alarm	RD Stufe 1, Z1	2	4 137 Pistc6 Pistc10 Pist3 PistcPeak t2	21 Pist <mark>24 Pist4 Pistc</mark> 6 Pistc <mark>17 Pist11</mark>	Pist15 Pist16 Pist26 Pist4 Pistc7 Pis	tcPeak t 22 Pist4
Event	06.09.2007 23:43:57	Alarm	KKB Stufe 3, Z4	F	FT RMS value F	FFT RMS value	10 RMS values	FFT RM:
Information	06.09.2007 23:42:04	Pattern	0)	9	Neues Zustands-Muster Nr. 25466 eingetragen			
Event	06.09.2007 23:41:59	Alarm	KKB Stufe 4, Z3	1	4 Absolute maximum	340		
Event	06.09.2007 23:41:59	Alarm	KKB Stufe 4, Z3 KKB Stufe 1, Z1	4	Hr-25 Absolut(30 RMS values 30 RMS v	values 25 Absolute ma29 Absolute	<mark>: ma 25 Absolute ma</mark> 29 Absolute ma	24 Absolute ma
Event	06.09.2007 23:39:59	Alarm	7ul Ct 2 72	+	₩-11 RMS values		11 RMS values	
Event	06.09.2007 23:35:02	Alarm	RD Stufe 2, Z2	1	← 5 Piston-rod position analysis		28 Piston-rod position analysis	
Event	06.09.2007 23:32:56	Alarm	RD Stufe 2, Z2 RD Stufe 1, Z1 RD Stufe 2, Z2 KKB Stufe 1, Z1	- 2	21 Piston-rod position analysis		12 Piston-rod position analysis	
Event	06.09,2007 23:31:57	Alarm	RD Stufe 2, Z2	2	4₀7 Piston-rod position analy:28 Piston-rod position analysis 28 Piston-rod position analysis 7 Piston-rod position analysis 7			
Event	06.09.2007 23:31:56	Alarm	KKB Stufe 1, Z1	4	y 30 Absc 33 Absolute30 Absolute <mark>9 Al</mark>	<mark>bsolute r</mark> 9 Absolute r <mark>30 Absolute</mark> FF	T RMS vaFFT RMS va 25 Absolute:	85 Absolute25 Al
Event	06.09.2007 23:30:57	Alarm	KKB Stufe 3, Z4	F	FT RMS value F	FFT RMS value	FFT RMS value	FFT RM:
Event	06.09.2007 23:26:57	Alarm	RD Stufe 1, Z1		17 Piston-rod position analysis	11 Piston-rod position analysis	3 Piston-rod Peak-Peak	17 Pisto
Event	06.09.2007 23:22:00	Alarm	RD Stufe 3, Z4	4	₩-2 Piston-rod Peak-Peak			
Event	06.09.2007 23:22:00	Alarm	KKB Stufe 1, Z1	¥ 4		naximum 30 RMS values	30 Absolute maximum 30 R	MS values

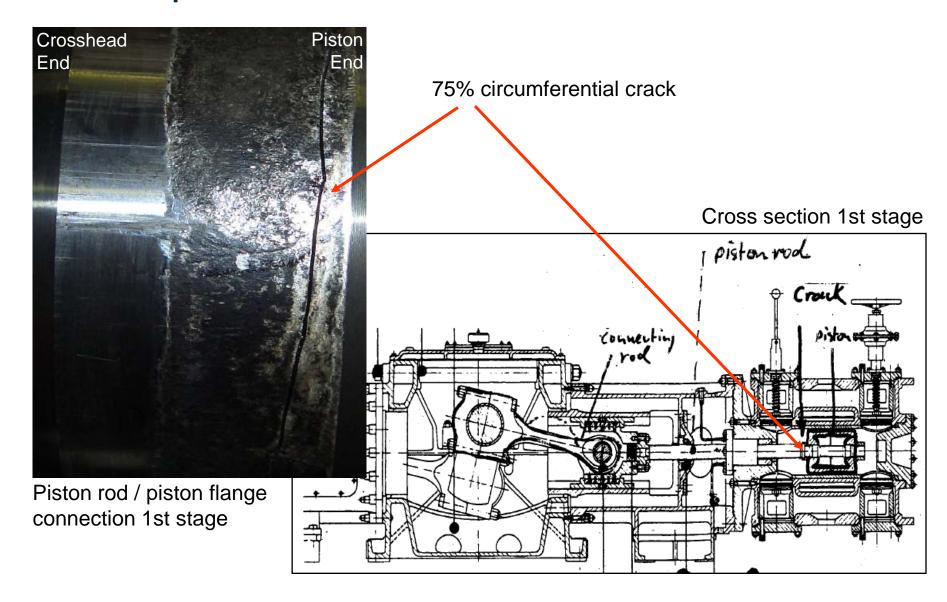
Data Analysis Crosshead Vibration & Rod Position Normal Signatures and at Trip Alarm



3D Trend Data Analysis Comparison: RMS Crosshead Vibration and Pk-Pk Rod Run Out



Findings Failed Component 2007



Findings

Comparison with 2002 Rod Failure

	2002	2007						
Shutdown Parameter	Vibration at CHS	Vibration at CHS & Piston rod position						
Shutdown Limits (% measure range)								
Vibration at Crosshead	+30 %, 36 segments	+8 %, 36 segments						
Piston rod position	NA	+35 %, 8 segments						
<u>Damages</u>	Complete rod breakBent con-rodRipped casing	 Only a partially cracked piston rod 						
Avoided damages	Total loss of machine frame	Complete piston rod breakBent con-rodRipped casingTotal loss of machine frame						
Duration of repairs	6 weeks	24 hours						

Findings Comparison of Economic Impact

- Since 2002, following the adoption of the online monitoring system, maintenance expenses for this compressor have decreased by: \$123,000 USD p.a.
- Production down time of 6 weeks resulting from the 2002 event could have exceeded 3 to 6 months without online monitoring.
- By increasing functionality of the MMS to include piston rod position production outage time was decreased to only
 24 hours
- Production losses due to the shutdown in 2007 compared with the unplanned shutdown 2002 were decreased by 43,400 mt urea

Summary / Lessons Learned

- Using crosshead vibration as a shut down parameter, modern reciprocating machinery protection systems can significantly reduce consequential damages resulting from piston rod failures.
- Using rod run out as a shut down parameter, it is possible to detect the development of a piston rod crack (possibly not detectable with vibration) and successfully trip, before it fails completely.
- Automated machine monitoring systems with rod position shutdown functions lead to lower cost of production and higher plant efficiency.

THANK YOU FOR YOUR ATTENTION!

QUESTIONS?