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Assessment and prediction of the thermal performance of a centralized latent heat thermal energy storage utilizing artificial neural network

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Outline

- Introduction
- LHTES Applications for buildings of the future
- Objectives
- Methodology
- Physical model
- Results

Motivation:

Why the centralized LHTES system?



Experimental investigation of energy saving in buildings with PCM cold storage



Advantages:

Experimentally proven that the stored cooling energy for about 3hr for air velocity of 1.5m/s.

<u>Disadvanťages:</u>

 Numerical model was not consistent with experimental data during transient period of phase change.

Stritih U, Butala V. Experimental investigation of energy saving in buildings with PCM cold storage. International Journal of Refrigeration. 2010;33:1676-83.

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Limitations of existing work

1- The heat transfer problem is simply formulated into twodimensional transient diffusion equation in most PCM problem.

2- The effect of thermal stratification and buoyancy-driven convection phenomena needs to be further investigated in LHTES.

3- Thermal behavior of phase change is not sufficiently investigated due to the removal of velocity convective term.



≻To develop a 3-D numerical model of LHTES and to study its thermal behavior under various conditions.

 \succ To validate the integrated model with the experimental data.

 \succ To carry out the parametric study to investigate the effect of the geometrical parameters on the HTF outlet air-temperature.

> To investigate the effect of integrated system on demand.

Methodology & Governing equations

- The enthalpy-porosity technique for modeling convectiondiffusion phase change is employed. The flow in solid–liquid region is modeled by the Darcy's law.
- The solver algorithm of coupling pressure-velocity is employed for solving momentum and continuity equations.
- VOF algorithm is used to update the volume fraction at each unit cell step by step in the entire computational domain.

Governing equations

Continuity

$$\frac{\partial u_n}{\partial t} + u_i \frac{\partial u_n}{\partial x_i} = 0$$
Momentum

$$\frac{\partial}{\partial t} (\rho \ u_i) + \frac{\partial}{\partial x_j} (\rho \ u_j u_i) = \mu \frac{\partial^2 u_i}{\partial x_j \partial x_j} - \frac{\partial p}{\partial x_i} + \rho g_i + S_i$$

$$S_i = -\frac{C(1-\gamma)^2}{\gamma^3 + \varepsilon} u_i$$

$$\rho_{(PCM)} = \frac{\rho_l}{\beta(T-T_l) + 1}$$

$$\gamma = \frac{T-T_s}{T_l - T_s}$$

$$\mu = 0.001 \times exp\left(A + \frac{B}{T}\right)$$

дα

дα

 α_n = the volume fraction of nth fluid in the computational cells.

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Governing equations

Energy

$$\frac{\partial}{\partial t}(\rho h) + \frac{\partial}{\partial x_i}(\rho u_i h) = \frac{\partial}{\partial x_i}\left(k\frac{\partial T}{\partial x_i}\right) + S_h$$
$$S_h = \frac{\partial(\rho\Delta H)}{\partial t} + div(\rho \underline{u}\Delta H)$$
$$H = h + \Delta H$$
$$\Delta H = F(T)$$
$$(L = T > T_{Vin})$$

 $F(T) = \begin{cases} L, & T \ge T_{liquid} \\ L(f), & T_{liquid} \ge T \ge T_{solid} \\ 0, & T < T_{solid} \end{cases}$ $\underline{u} = \begin{cases} u_i, & \text{for liquid region} \\ (f)u_i, & \text{for mushy region} \\ 0, & for soild region \end{cases}$ $\underline{u} = -\left(\frac{K}{\mu}\right) gardP$

Governing equations

Boundary and initial conditions:

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Symmetry boundary conditions at side:

$$\frac{dT}{dx}\Big|_{x=0} = \frac{dT}{dx}\Big|_{x=L} = 0,$$
$$\frac{dT}{dy}\Big|_{y=-H/2} = \frac{dT}{dy}\Big|_{y=H/2} = h\left(T_{out,i} - T_{in,i}\right)$$

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Initial condition:

$$t = 0, \quad T = T_{i} = 288 \text{K}$$

 $u = v = 0, \quad w = 1.5 \text{m/s}$



3-D LHTES-proposed model



3-D LHTES-developed model characterisation





The liquid fraction and phase distribution of PCM PCM temperature contours for the evolution of melting process at case 1 at t = 1.5hr

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3-D LHTES-developed model validation



Temperature of the inlet air $T_i = 26^{\circ}$ C

Performance of LHTES

Assessment long-term performance of LHTES utilizing ANN



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Integrating the centralized LHTES system into a building model

- TRNSYS Building thermal response for single-zone model of 11.44m×5.69m×2.76m
- LHTES thermal response function
- Ventilation system modes
- Scheduled night ventilation
- Characterization and optimal design of LHTES system

Integrating the centralized LHTES system into a building model



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Performance evaluation of an integrated unit with energy building model



Indoor Air Temperature Histories With and Without LHTES System Combined With Night Ventilation For (6–9) Days of July

Thanks for your attention